

# *Computational Aspects of Modeling Fluid Flow in Micro-junctions*

*OU Supercomputing Symposium*

*Oct. 7, 2008*

Evan Lemley & Tim Handy

Department of Engineering and Physics

University of Central Oklahoma

---

---

# *Overview*

Background / Current Research

Problem Description

2D Simulations

3D Simulations Mark I

3D Simulations Mark II

Conclusions and Future Work



# *Applications*

Flow in microfluidic flow networks and flow in porous networks are of interest in many engineering applications.

Applications include porous media, micro-power generation, biomedical, computer chips, chemical separation processes, micro-valves, micro-pumps, and micro-flow sensors

Yanuka, M., Dullien, F.A.L., and D.E. Elrick, 1986, "Percolation Processes And Porous Media I. Geometrical And Topological Model Of Porous Media Using A Three-Dimensional Joint Pore Size Distribution," J.Colloid Interface Sci., **112**, pp. 24-41.

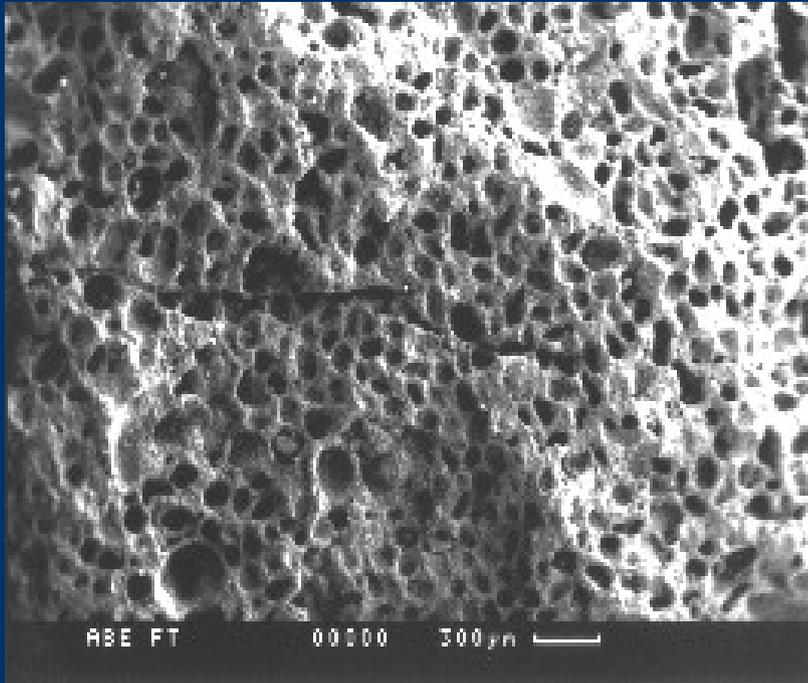
Lee, W.Y., Wong, M., and Zohar, Y., 2002, "Microchannels in Series Connected Via a Contraction/expansion Section", J. Fluid Mech., **459**, pp.187-206.

Judy, J., Maynes, D., and Webb, B.W., 2002, "Characterization of Frictional Pressure Drop for Liquid Flows Through Microchannels," Intl. J. Heat Mass Trans., **45**, pp.3477-3489.

**Flow in these applications is usually laminar**

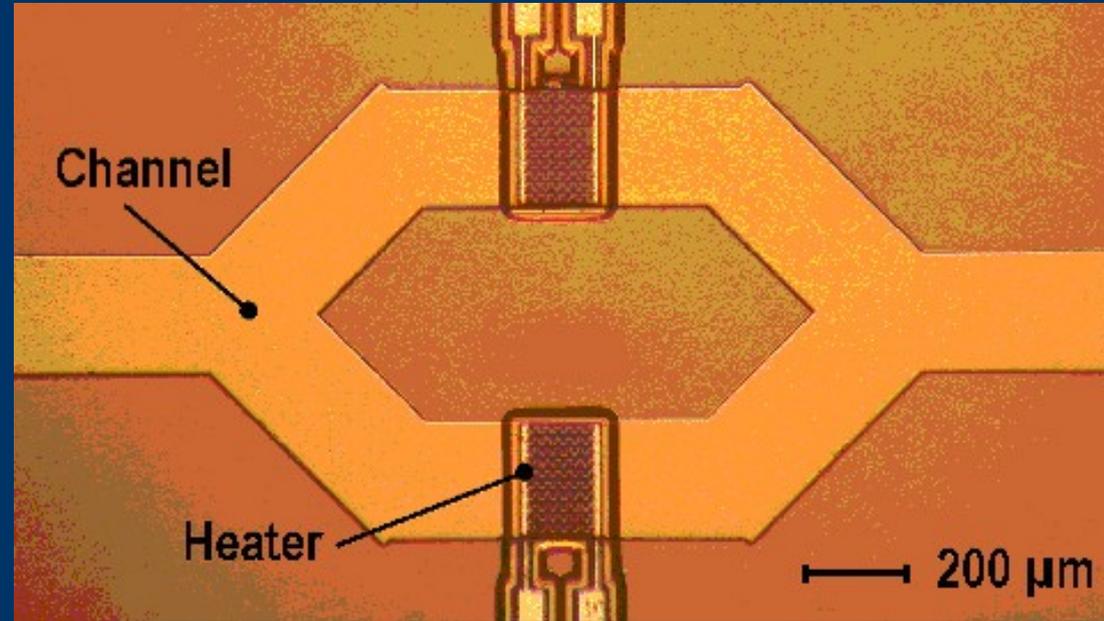
Graveson, P., Branbjerg, J., and Jensen, O.S., 1993, "Microfluidics a Review," J. Micromech. Microeng., **3**, pp.168-182.

# *Porous Media*



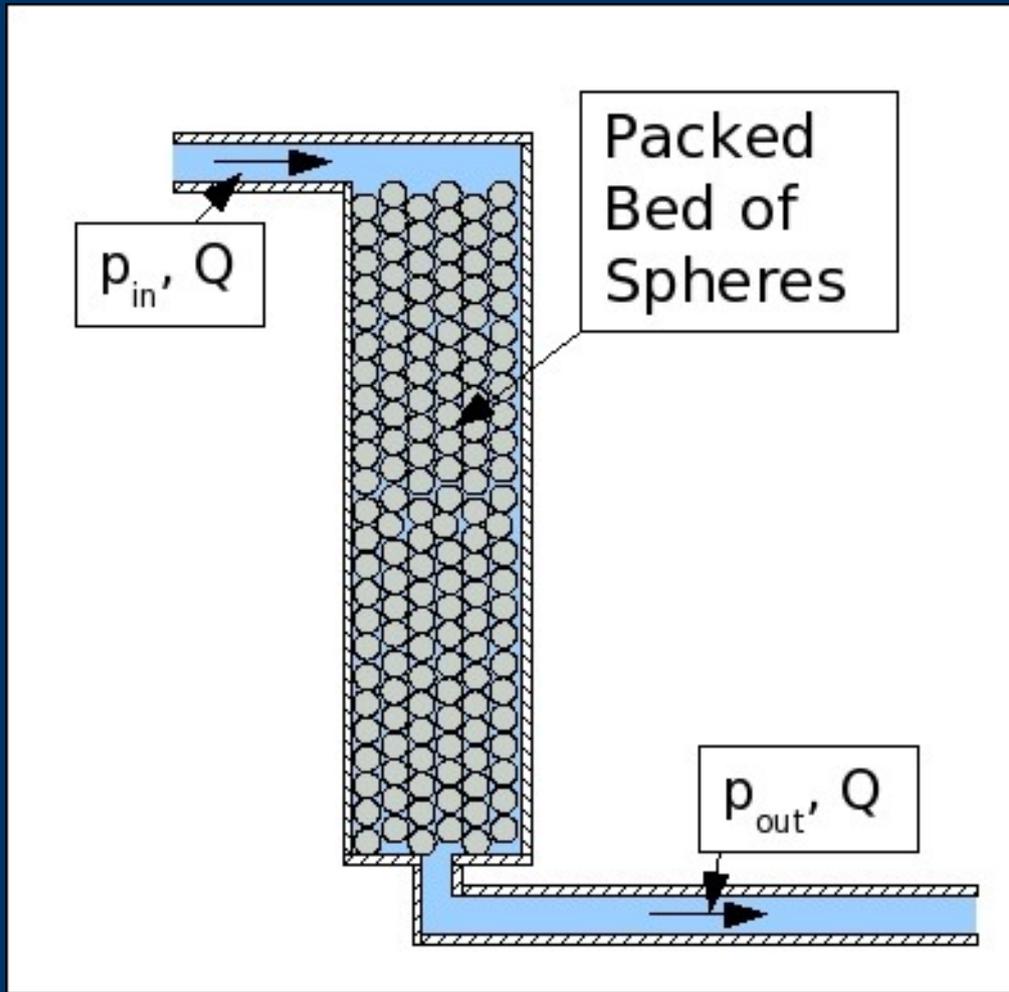
Highly porous magnesian limestone.  
([www.dawntnicholson.org.uk](http://www.dawntnicholson.org.uk))

# *Microfluidic Devices*



Microfluidic Valve Structure.  
(<http://www.cchem.berkeley.edu/sjmgrp/people/boris/boris.htm>)

# Artificial Porous Media



Packed Beds, Gas and Liquid Filters  
Sphere sizes  $\mu\text{m}$  to  $\text{cm}$   
Hold-up for chemical reaction, thermal processing, or filtering

## *Current Research*

Collaborative Effort with Dimitrios Papavassiliou and Henry Neeman from OU (began Fall 2004)

Simulation of Flow of Fluids through Porous Media

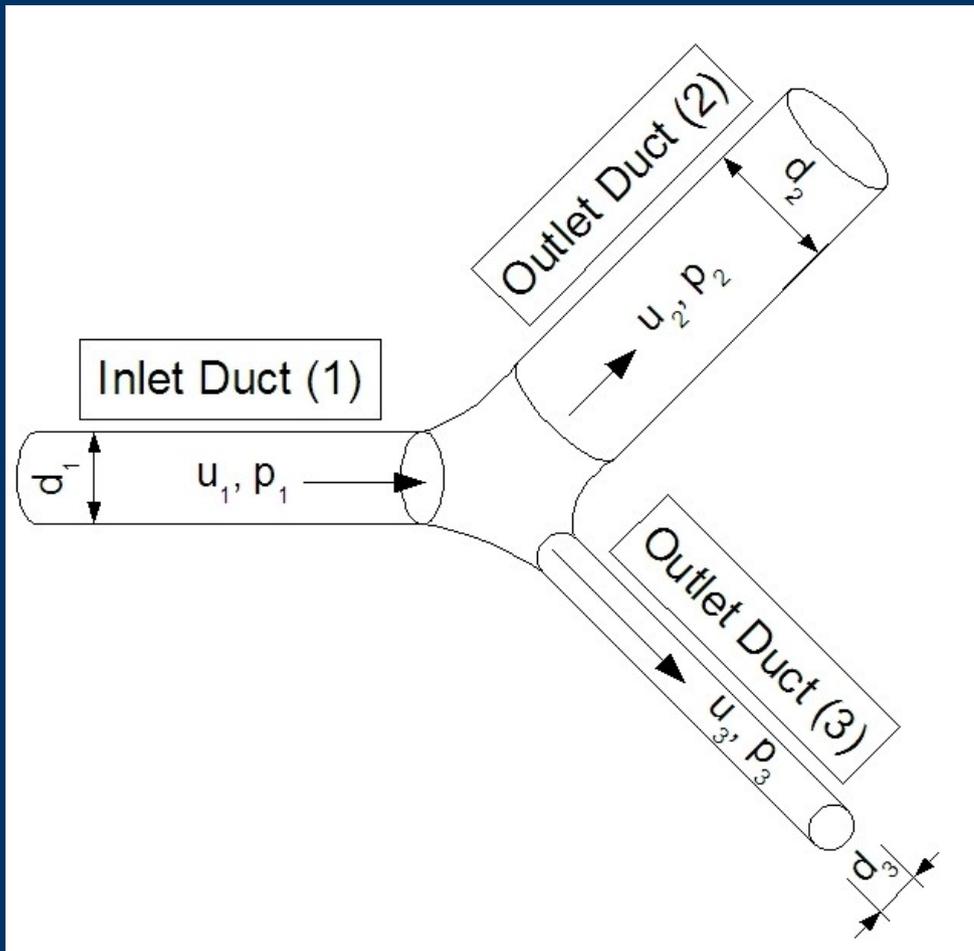
Code FTPM – Flow Through Porous Media.

*Solves for velocity and pressure at pore junctions in a randomly generated pore network.*

---

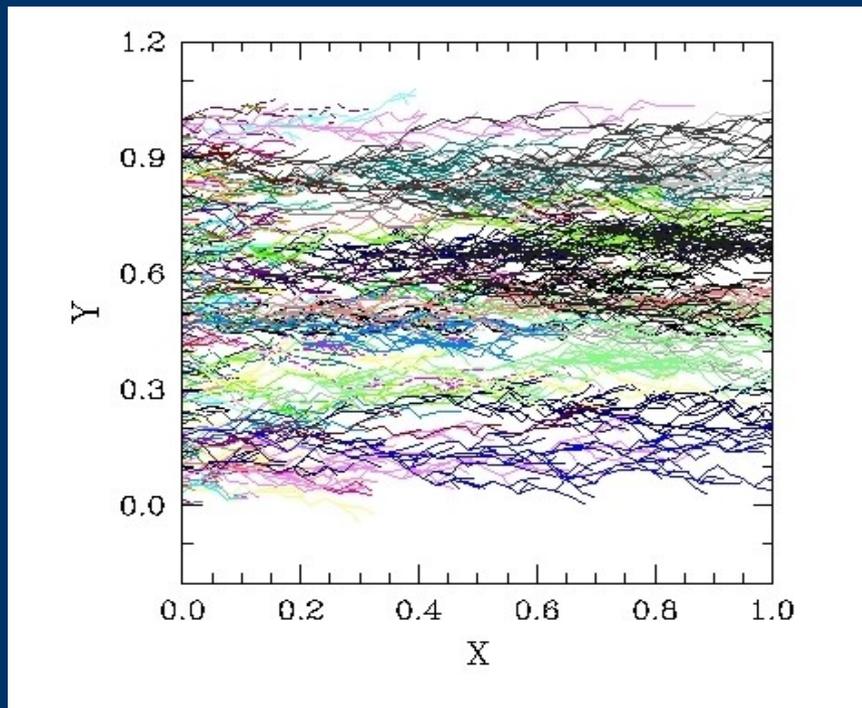
---

# Flow Network Analysis



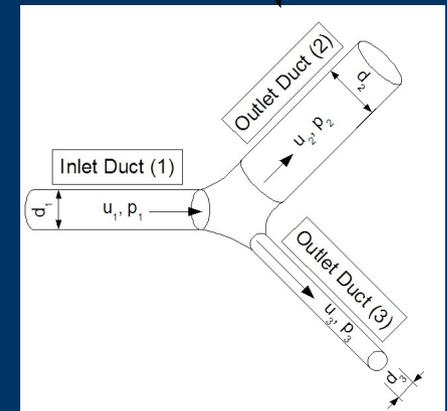
Design and Analysis of networks depends on knowledge of flow and energy losses in arbitrary branches. No systematic studies to generalize these bifurcations

# Porous Network Simulator (Collaboration with Univ. of Oklahoma)



Projection on the xy plane of a 3D network that has 200 entry points at  $x=0$ , porosity equal to 10% and a range of  $\pm 60^\circ$  relative to the x axis and  $\pm 30^\circ$  relative to the y axis.

3D Monte Carlo networks from normal, beta, or empirical distribution (pore size pdf)  
Coordination Number (1, 2, 3)  
number of pores entering and leaving a junction  
 $\theta \pm 90^\circ$



# Literature

T's and Y's – limited configurations and most are for turbulent flow

Basset, M.D., Winterbone, D.E., and Pearson, R.J., 2001, “Calculation of Steady Flow Pressure Loss Coefficients for Pipe Junctions,” Proc. Instn. Mech. Engrs., Part C, Journal of Mechanical Engineering Science, **215** (8), pp. 861-881.

W.H. Hager, 1984, “An Approximate Treatment of Flow in Branches and Bends,” Proc. Instn. Mech. Engrs., Part C, Journal of Mechanical Engineering Science, **198**(4) pp. 63-9.

Blaisdell, F.W., and Manson, P.W., 1967, “Energy loss at pipe junctions,” J. Irrig. and Drainage Div., ASCE, **93**(IR3), pp. 59-78.

Schohl, G.A., 2003, “Modeling of Tees and Manifolds in Networks,” *Proceedings of the 4<sup>th</sup> ASME/JSME Joint Fluids Engineering Conference*, **2**, Part D, pp. 2779-2786.

Bassett, M.D., Pearson, R.J., and Winterbone, D.E., 1998, “Estimation of Steady Flow Loss Coefficients for Pulse Converter Junctions in Exhaust Manifolds,” *IMEchE Sixth International Conference on Turbocharging and Air Management Systems*, IMechE HQ, London, UK, **C554/002**, pp.209-218.

Ruus, E., 1970, “Head Losses in Wyes and Manifolds,” J. Hyd. Div., ASCE, **96**(HY3), 593-608.

Laminar loss coefficients and elbows, reductions, contractions –  
much larger loss coefficients than turbulent case – strong dependence on Reynold's number.

Edwards, M.F., Jadallah, M.S.M., and Smith, R., 1985, “Head Losses in Pipe Fittings at Low Reynolds Numbers,” Chem. Engr. Res. Des., **63**(1), pp. 43-50.

Importance of roughness at microscale

---

---

# Problem Description

## Stagnation Loss Coefficient

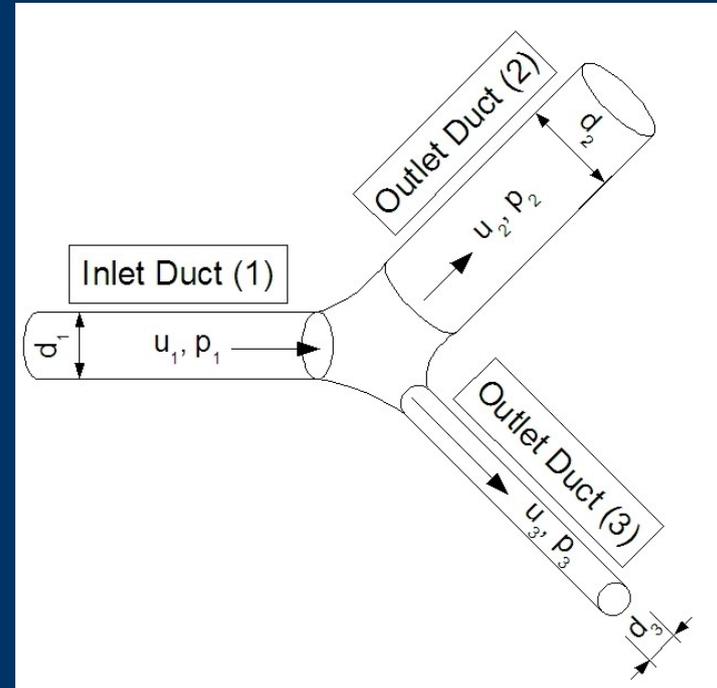
$$K_2 = \frac{\left[ \left( \frac{p_1}{\rho} + \frac{u_1^2}{2} \right) - \left( \frac{p_2}{\rho} + \frac{u_2^2}{2} \right) \right]}{\frac{u_1^2}{2}}$$

Parameters:

$d_2/d_1, d_3/d_1$

$\theta_2$  and  $\theta_3$

$f_2$  (this sets  $f_3$ ) – (why? other literature and possibility of simulations where this is unknown initially)



# *Automation of Geometry Generation and CFD Runs*

Custom Code was written to:

- (a) create GAMBIT journal files that instantiate the desired geometry based on existing 2D geometries.
  - (c) create a script that loads journal files into GAMBIT and meshes
  - (d) create all necessary preprocessing files for FLUENT.
  - (e) create post-processing files for FLUENT results and to tabulate results for a complete set of runs
- 
-

# Solution Methodology

## 2D Geometry Generalization

$$L_1 = 5 d_{\max}$$

$$L_2 \text{ and } L_3 = 10 d_{\max}$$

If  $d_2 > d_{\text{avg}}$ ,

then  $r_2 = 3d_2$ ;

else  $r_2 = 2d_2$

If  $d_3 > d_{\text{avg}}$ ,

then  $r_3 = 3d_3$ ;

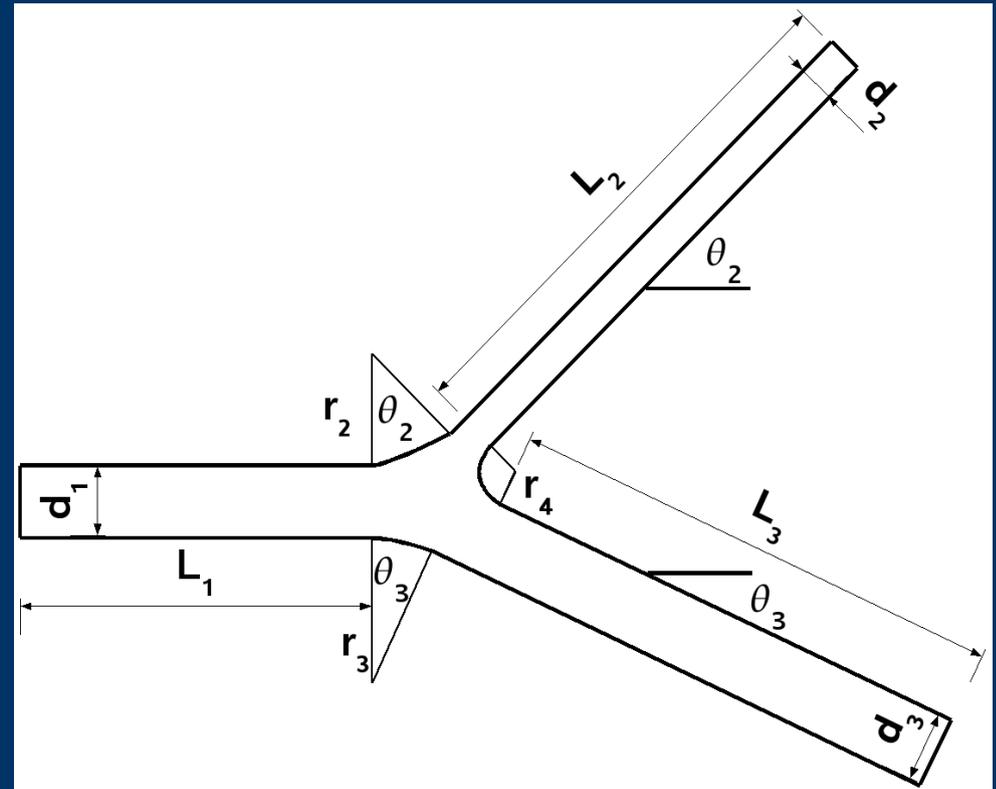
else  $r_3 = 2d_3$

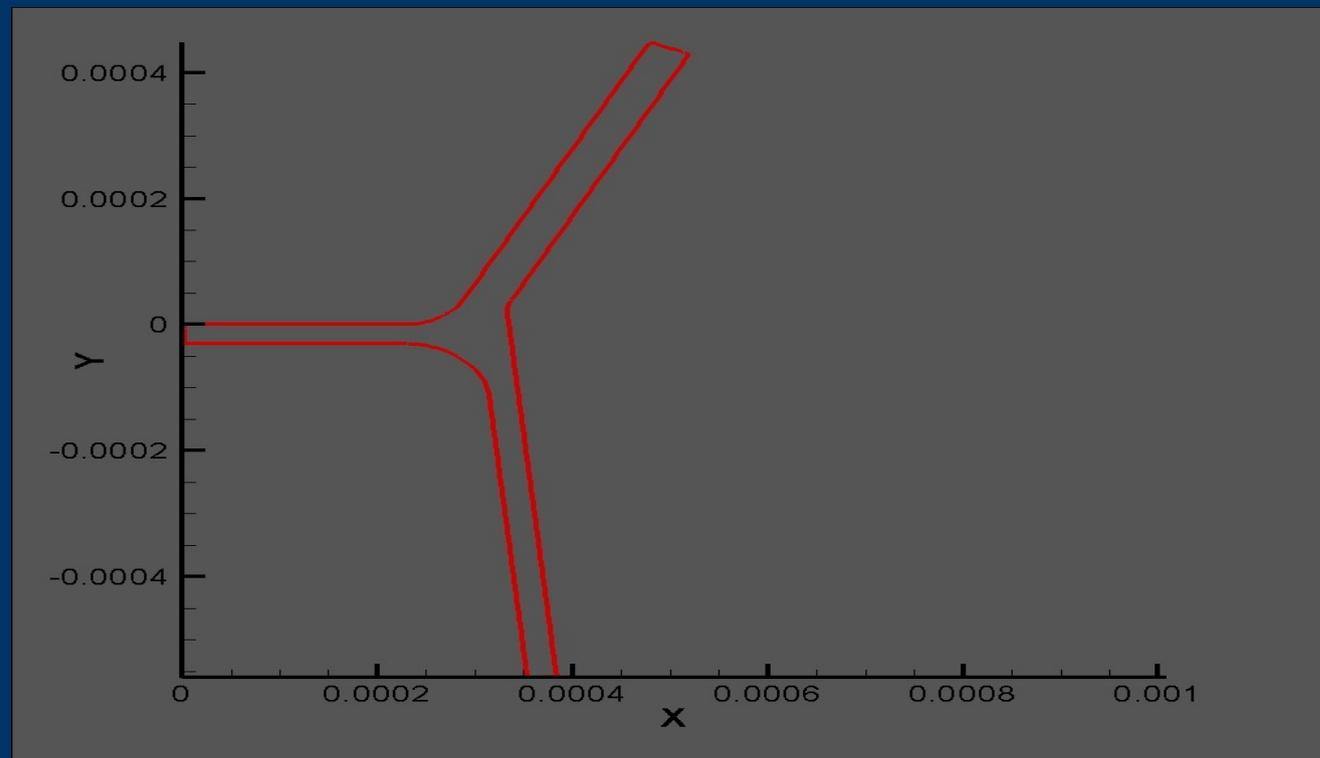
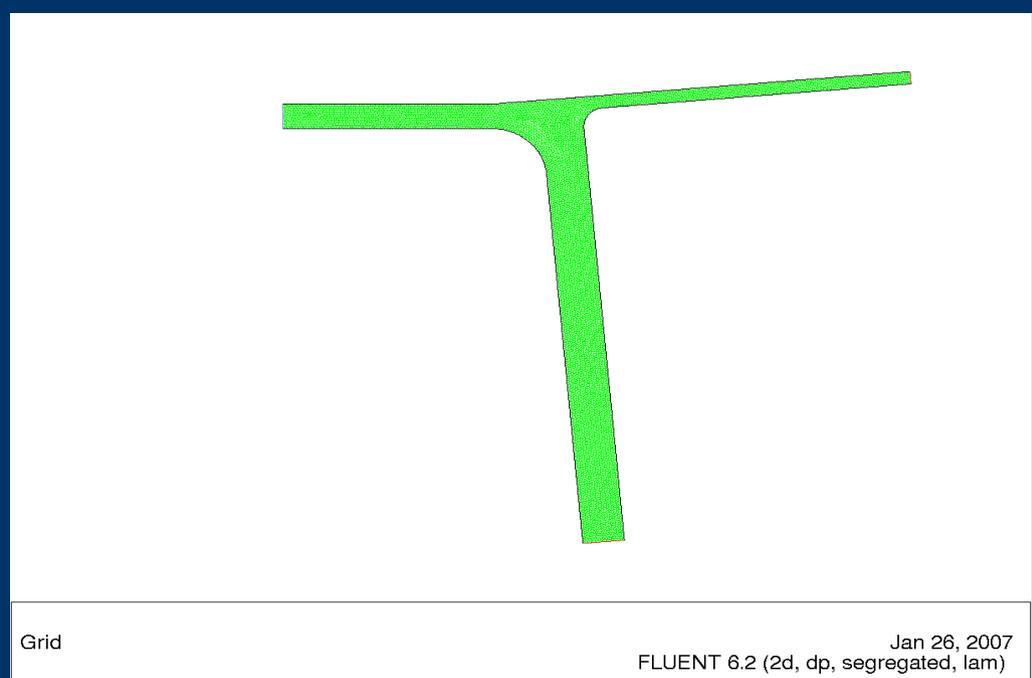
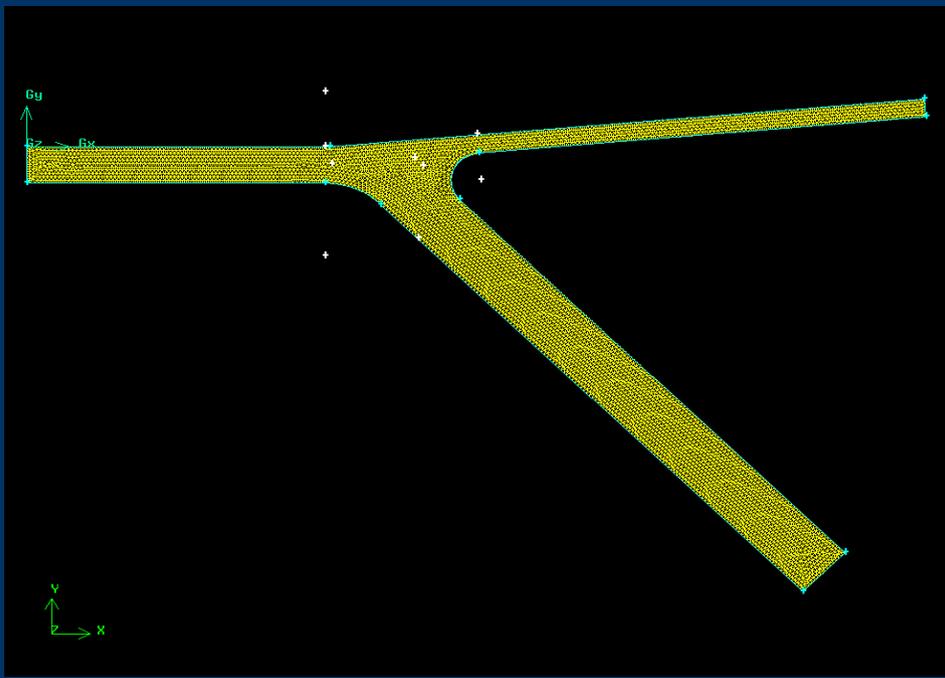
$$r_4 = d_{\max} / 2$$

Generalized Geometry

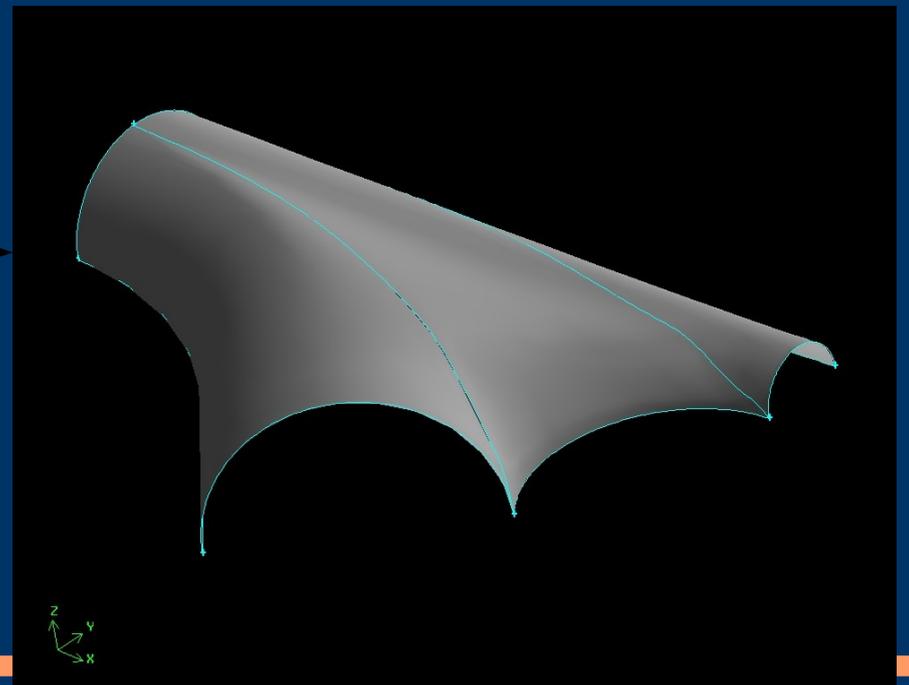
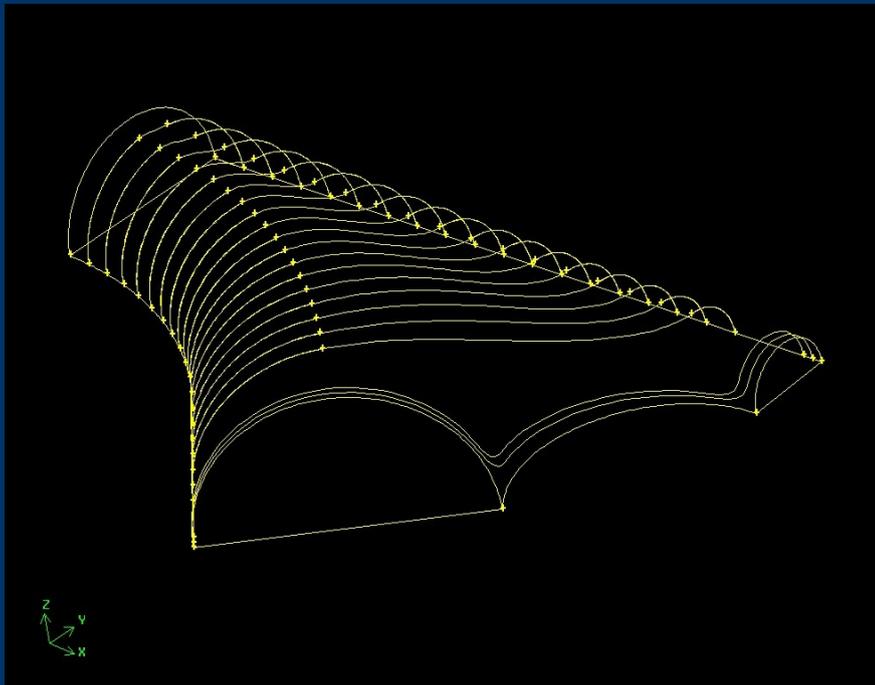
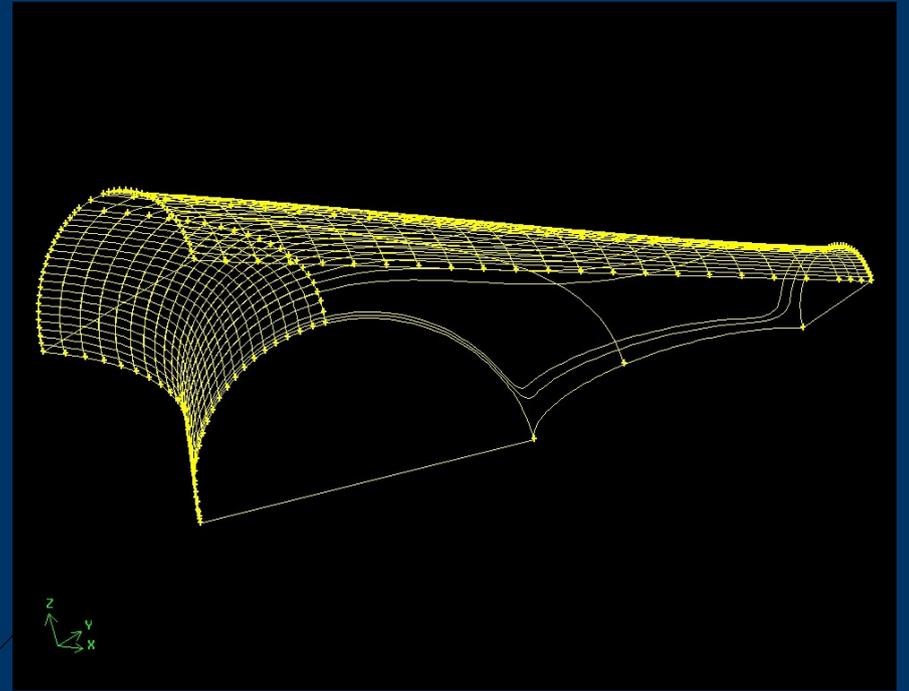
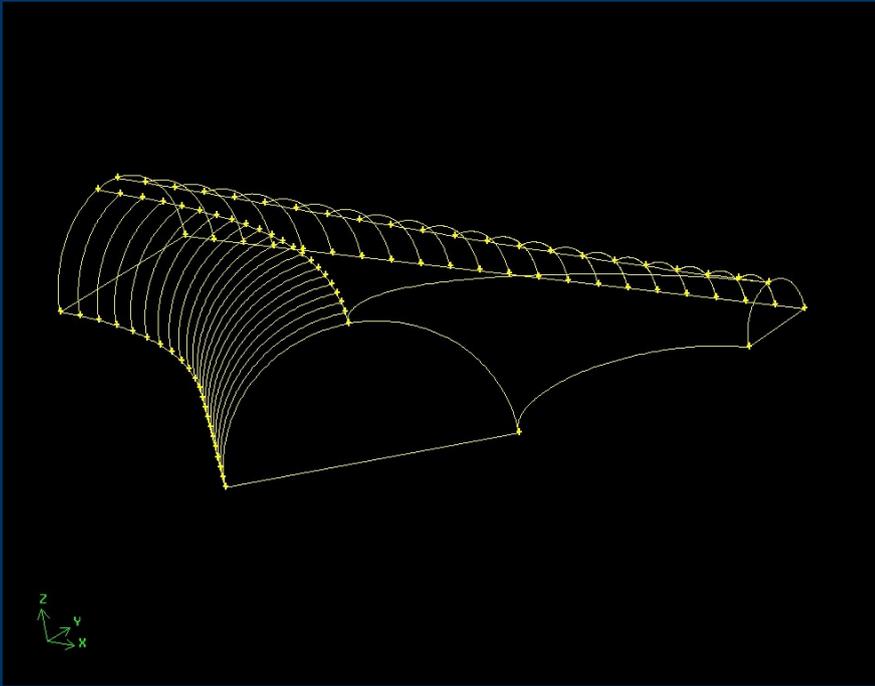
larger and smaller outlet ducts ; 2Dim. - 3Dim are underway

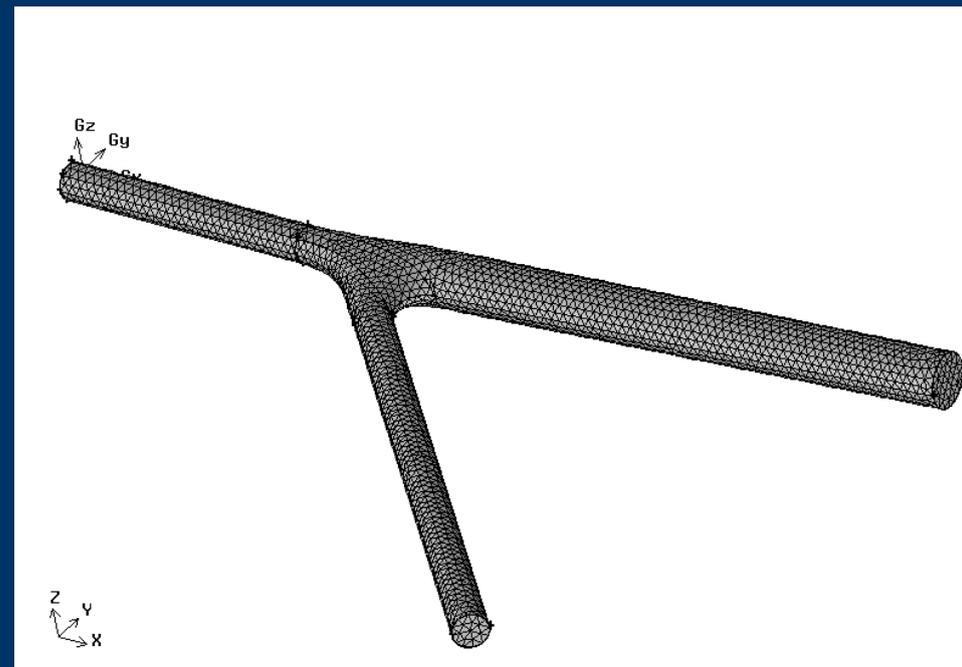
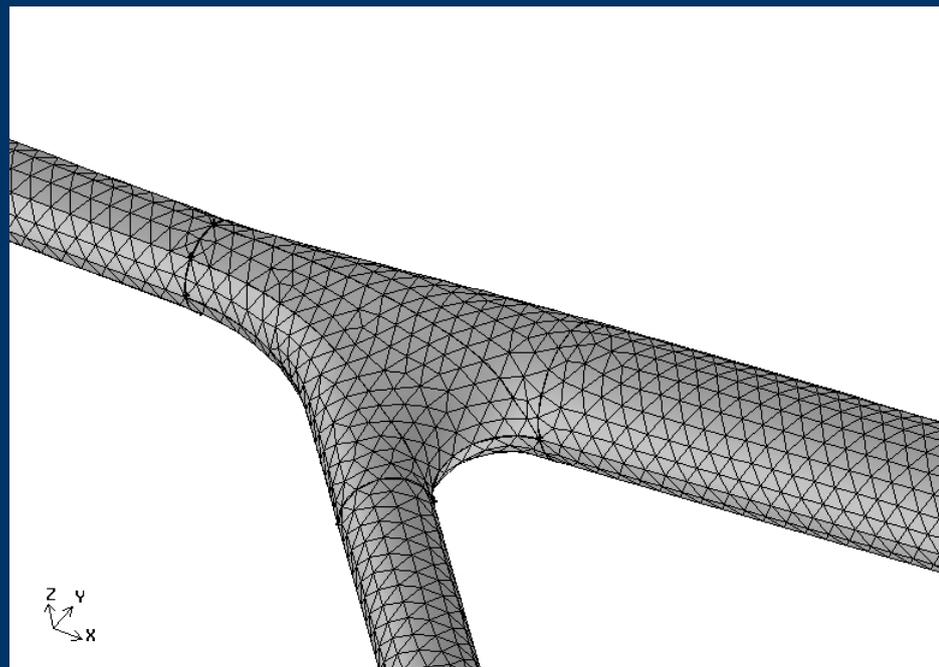
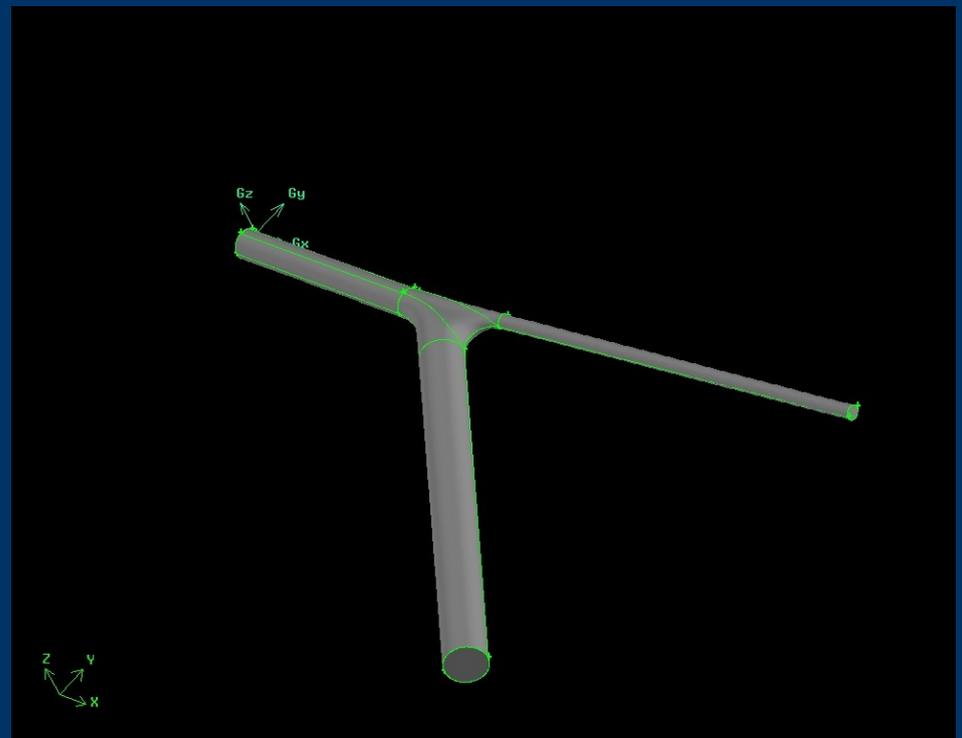
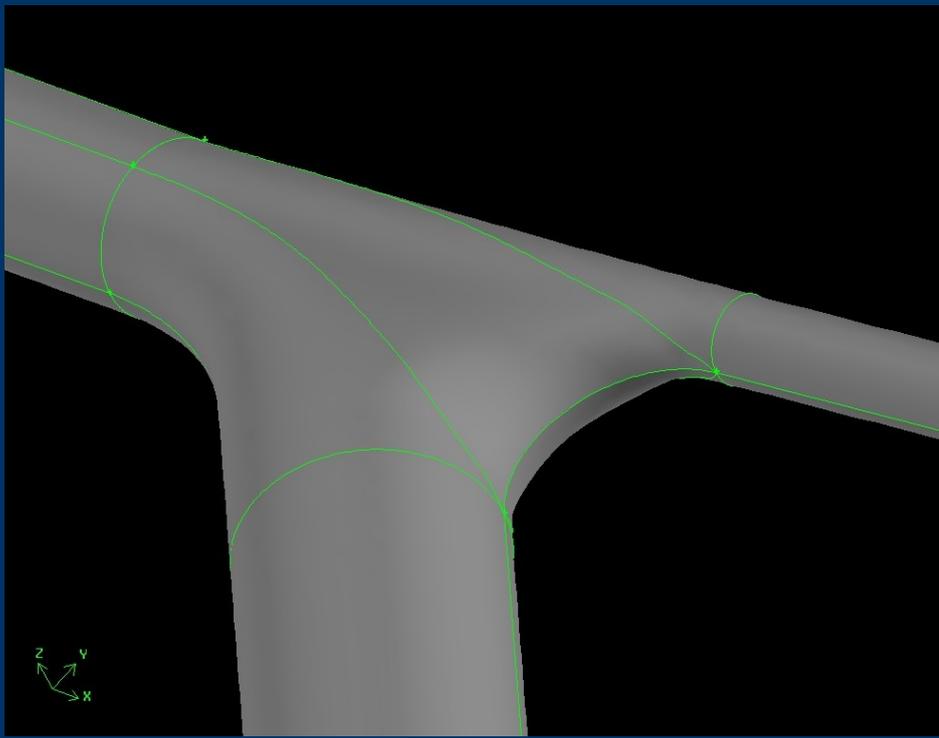
avoid sharp edges ; 5 – 90 degrees for angles





# 3D Geometry – Mark I





# *Simulation Parameters*

$Re_D$  was maintained at constant value at the inlet duct

$d_1$  was 30 microns. The fluid was chosen to be liquid water at 20°C. The inlet flow velocity,  $u_1$ , was set to 0.5 m/s giving a Reynolds number of 15

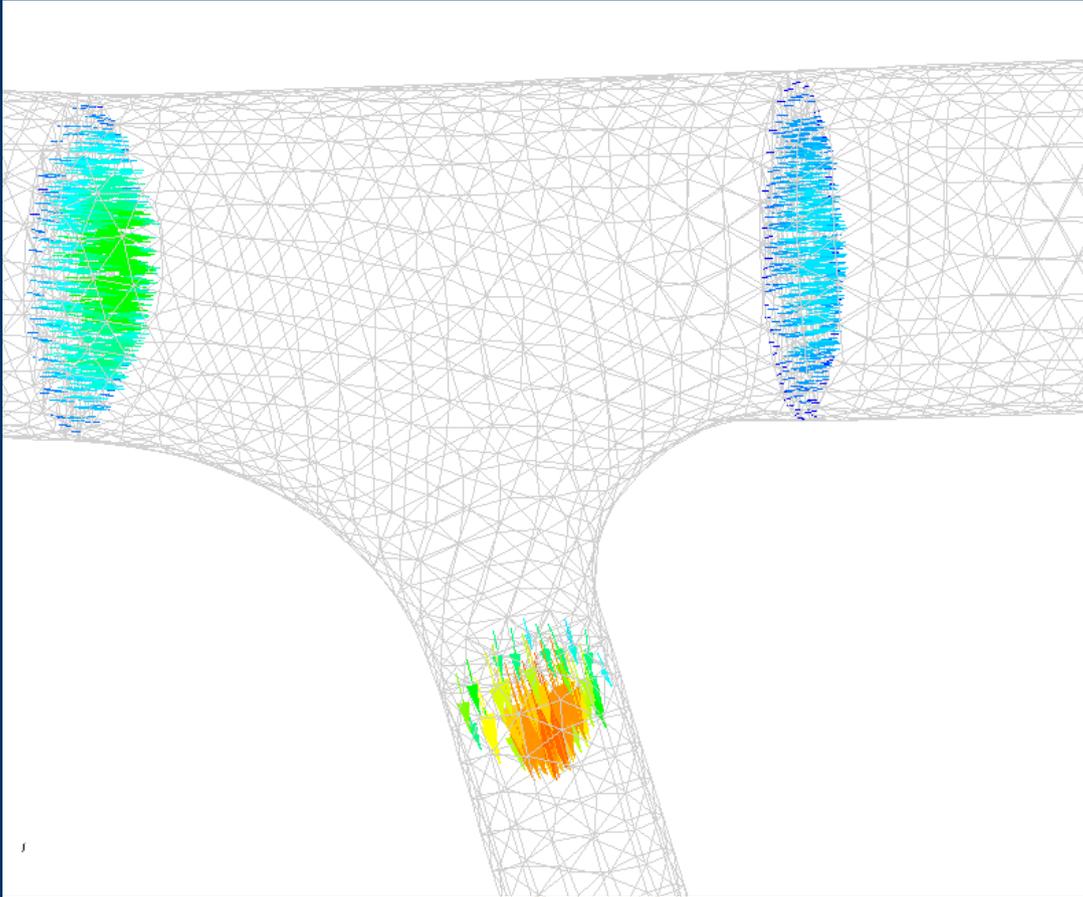
$Le_D = 0.06 Re_D D$  --- gives 0.9 D for  $Re_D = 15$

FLUENT output files contain surface averaged static pressure and magnitude of flow velocity at duct cross-sections at the duct inlets and exits.

---

---

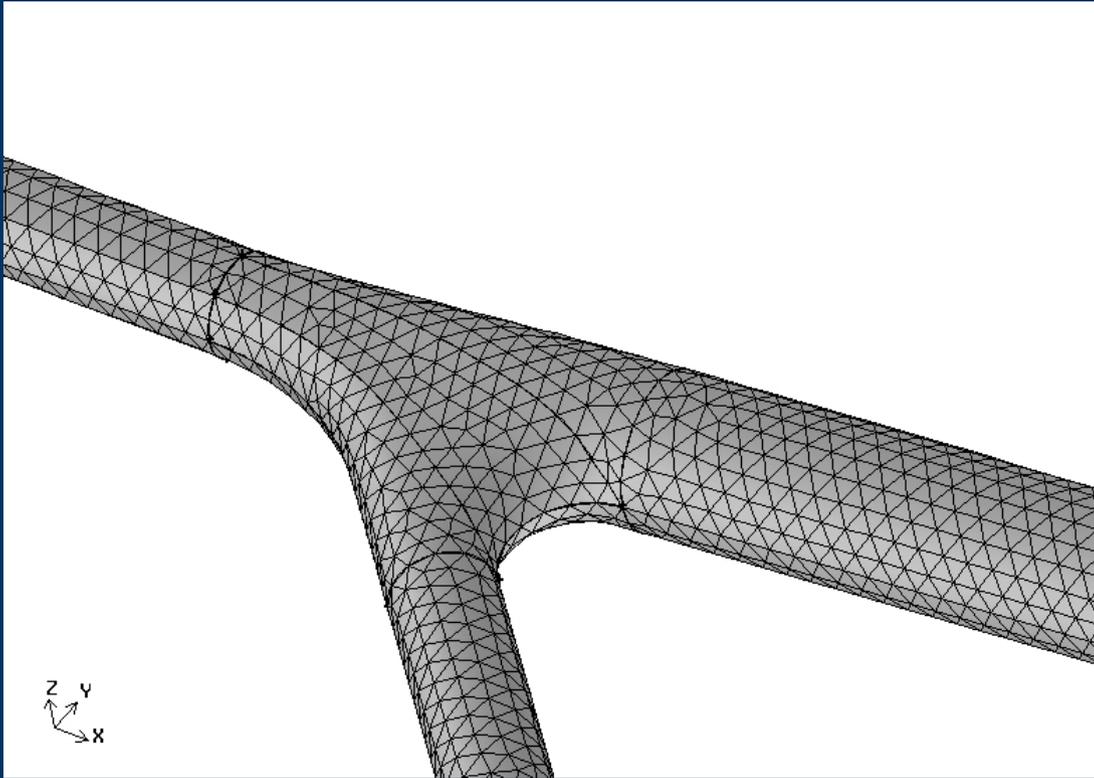
# *Duct Inlet/Outlets*



Duct inlet and exit sections considered to be where geometry of duct is the same as the downstream portion for outlets and upstream portion for inlet.



# Mesh Automation



Mesh was set to 1/4 of smallest duct

Tetrahedral Mesh

Large number of tests to assess ability to generalize the mesh (1/4 factor determined in this manner)

Some testing to verify no change in results with change in mesh size.

Inlet was specified as velocity inlet

Outlets were outflow boundaries – allowed specification of flow fraction

# *Numerical Methods*

Finite Volume solution of integral Navier Stokes

Steady-State 3D

Implicit

SIMPLE for pressure velocity coupling

1st order upwind scheme of momentum

discretization

Max number of iterations

Convergence criterion = 0.1%

---

---

## *Parameter Values*

$d_2/d_1$  and  $d_3/d_1 = 0.5, 1.0, 1.5$

$f_2 = 0.1, 0.3, 0.5, 0.7, 0.9$

$\theta_2$  and  $\theta_3 = 5^\circ, 25^\circ, 45^\circ, 65^\circ, 85^\circ$

600 runs attempted – 475 completed (*geometry issues on remainder*)

Suite of C++ procedures to create geometries, input files, read and collate results

Create GAMBIT script to create geometries

Create input files for GAMBIT and FLUENT

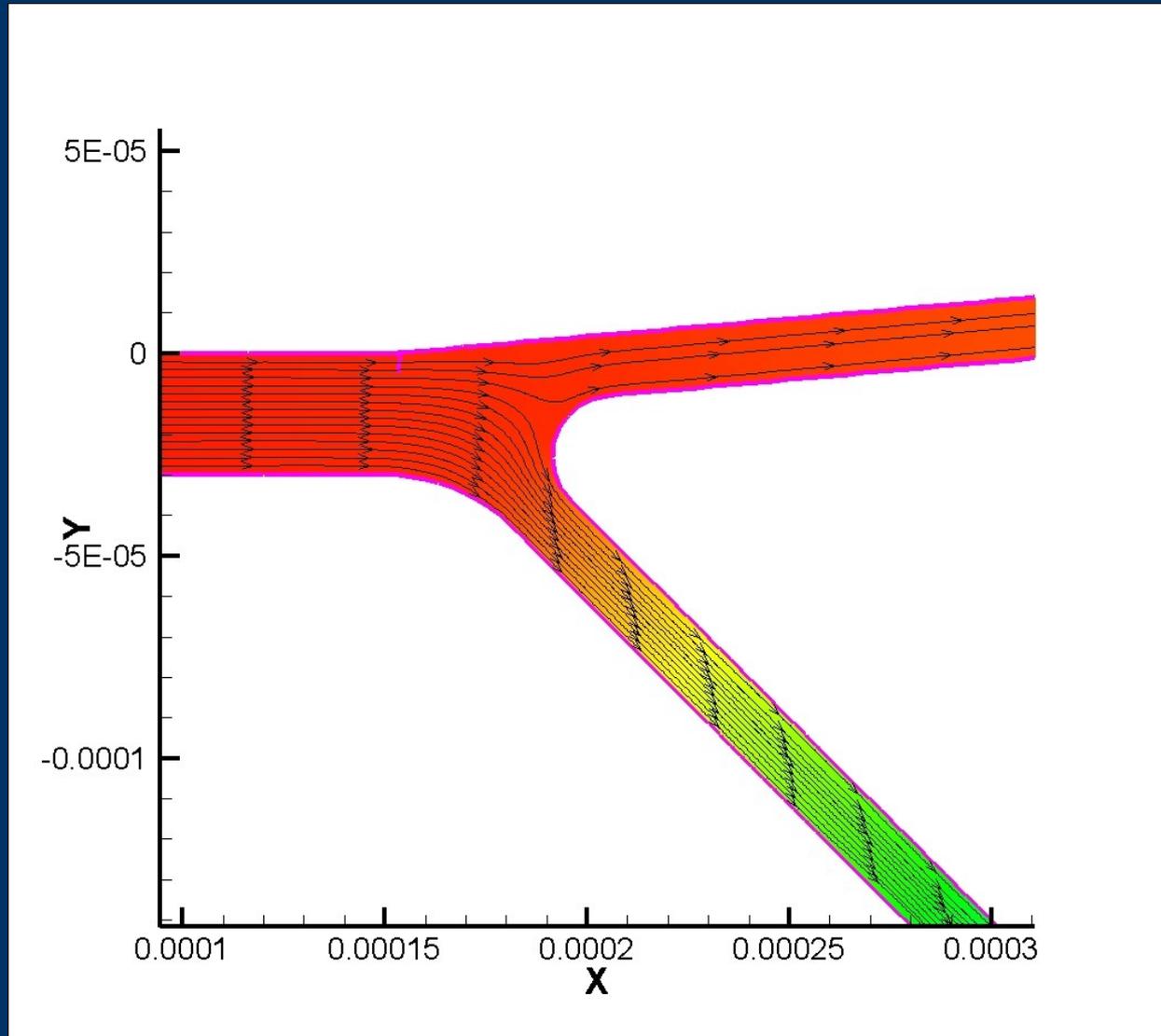
Read results files for static pressures and velocities averaged over surfaces in and out of junctions.

---

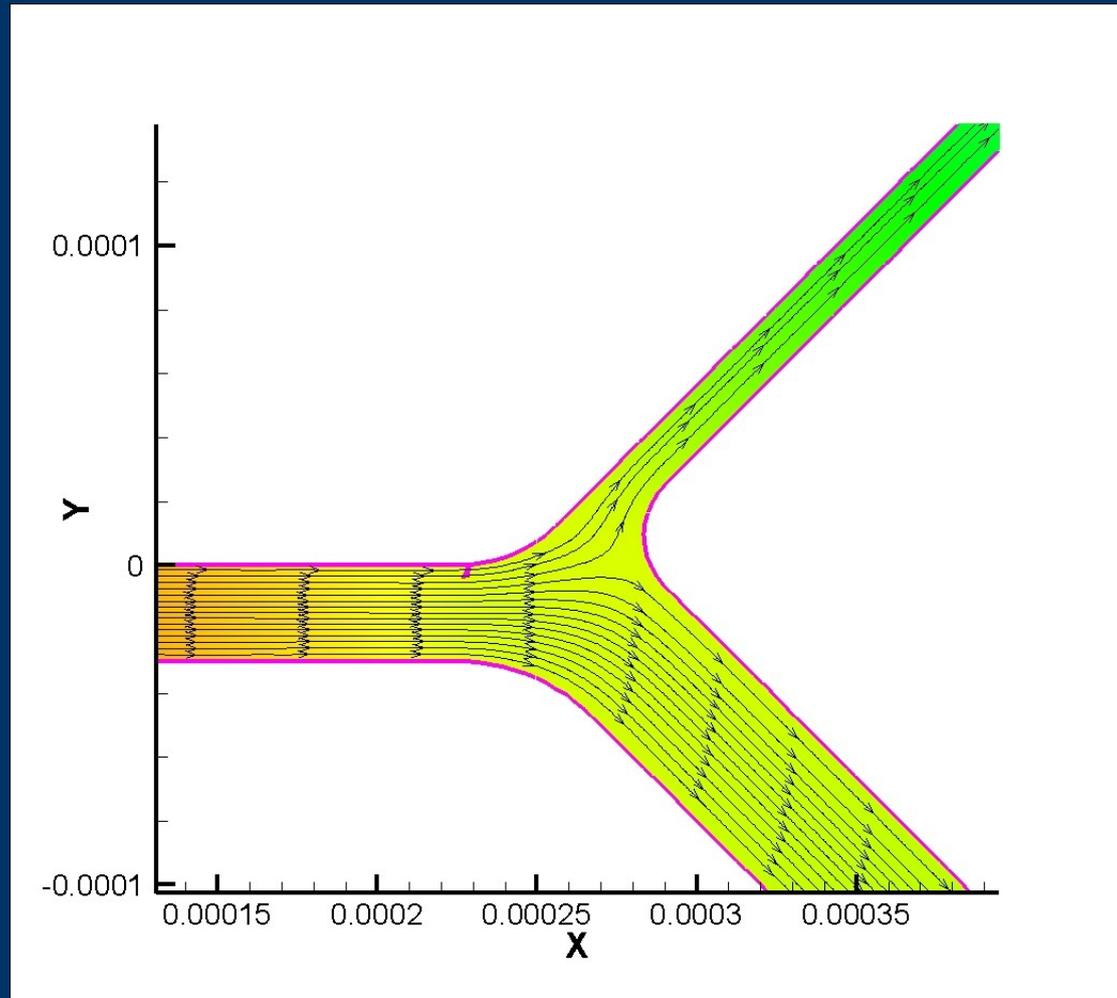
---

# Fluent Result

$f_2 = 0.1$ ,  $d_2/d_1 = 0.5$ ,  $\theta_2 = 5^\circ$ ,  $d_3/d_1 = 0.5$ , and  $\theta_3 = 45^\circ$

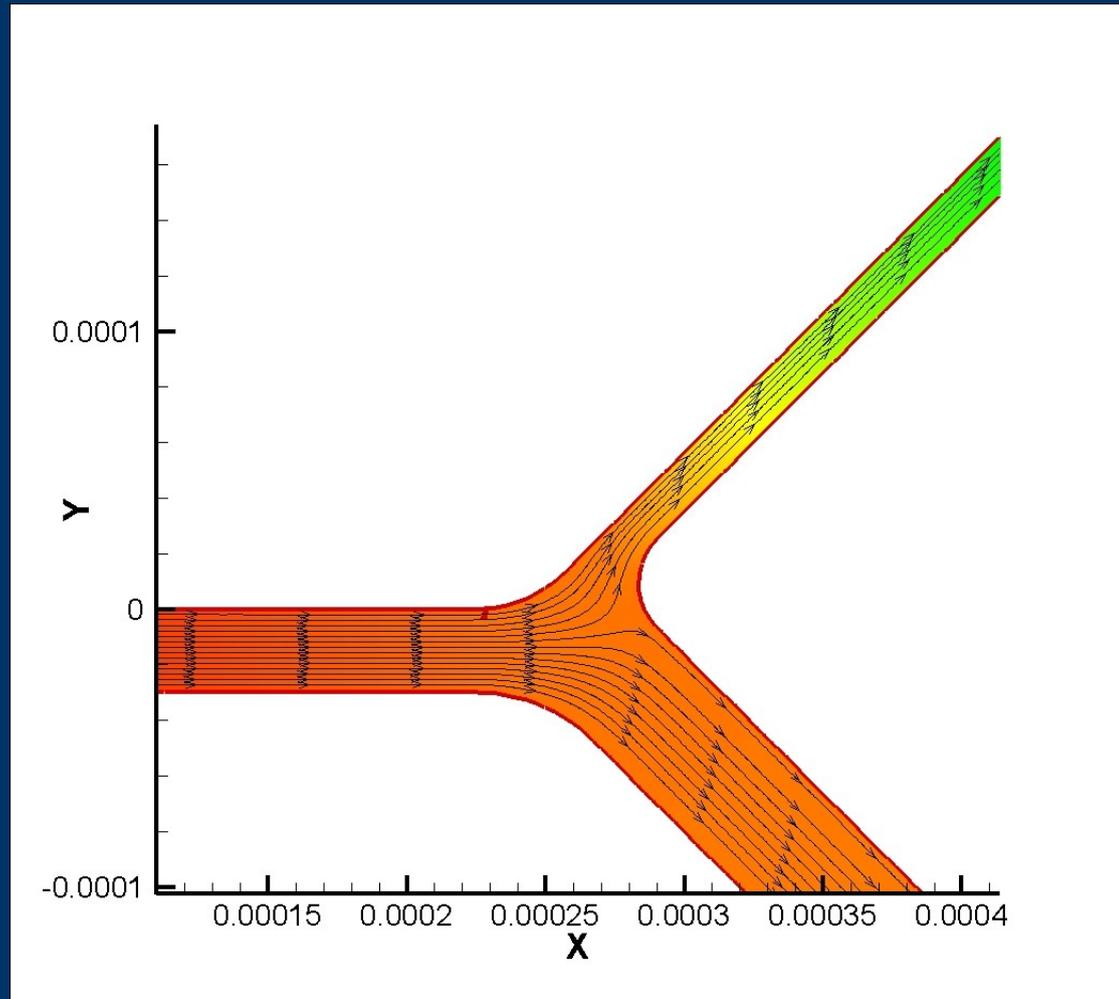


$f_2 = 0.1, \theta_2 = 45^\circ, \theta_3 = 45^\circ, d_2/d_1 = 0.5, d_3/d_1 = 1.5.$



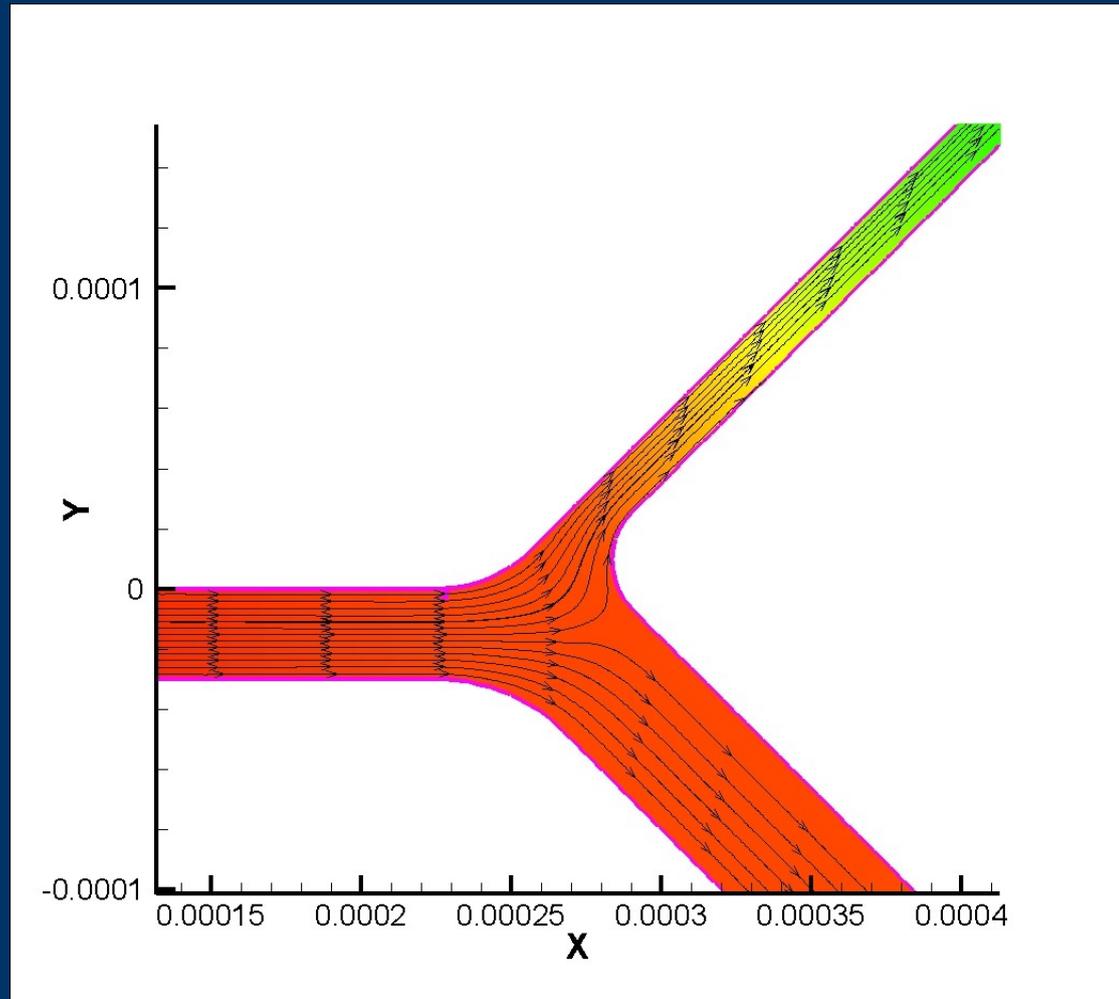
$$K_2 = 5.47$$

$f_2 = 0.3, \theta_2 = 65^\circ, \theta_3 = 45^\circ, d_2/d_1 = 0.5, d_3/d_1 = 1.5.$



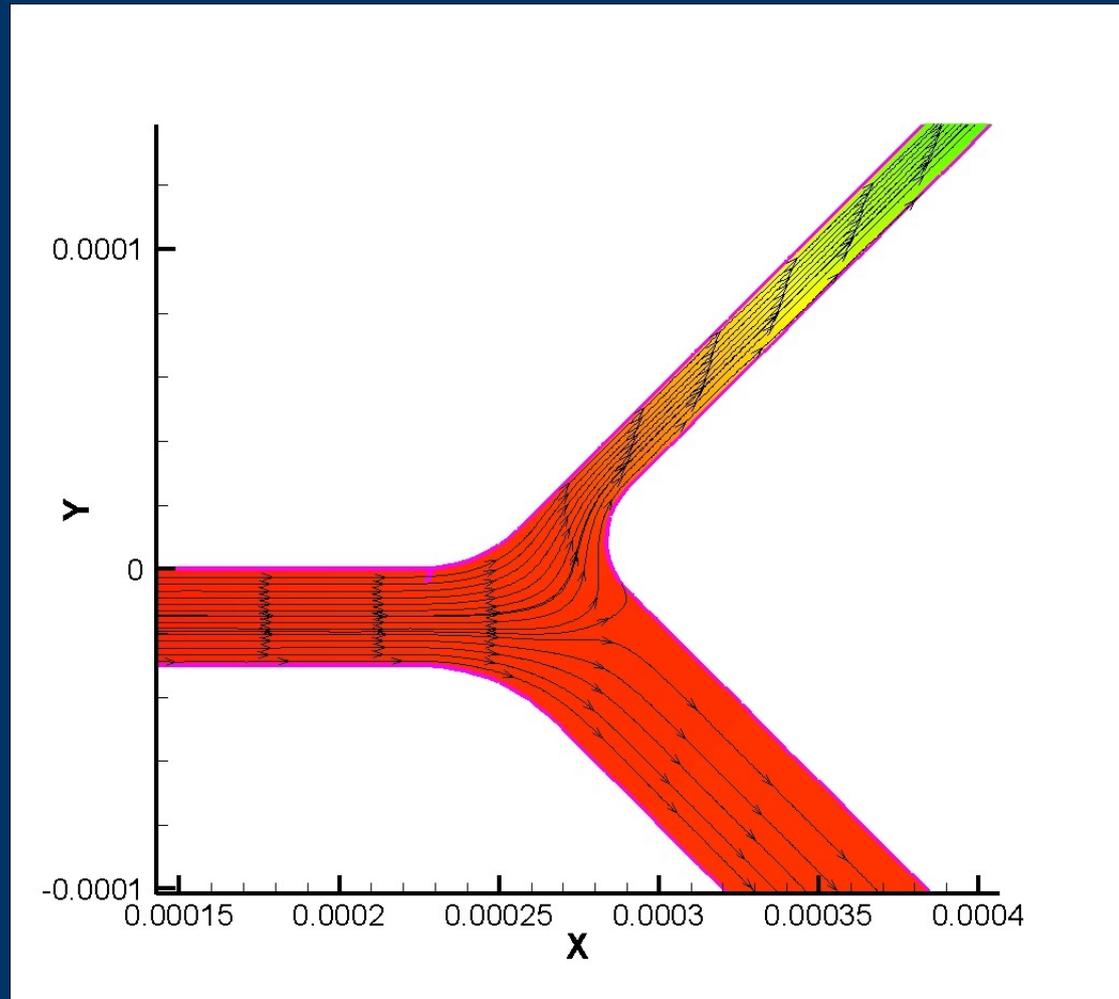
$K_2 = 11.6$

$f_2 = 0.5$ ,  $\theta_2 = 65^\circ$ ,  $\theta_3 = 45^\circ$ ,  $d_2/d_1 = 0.5$ ,  $d_3/d_1 = 1.5$ .



$K_2 = 18.4$

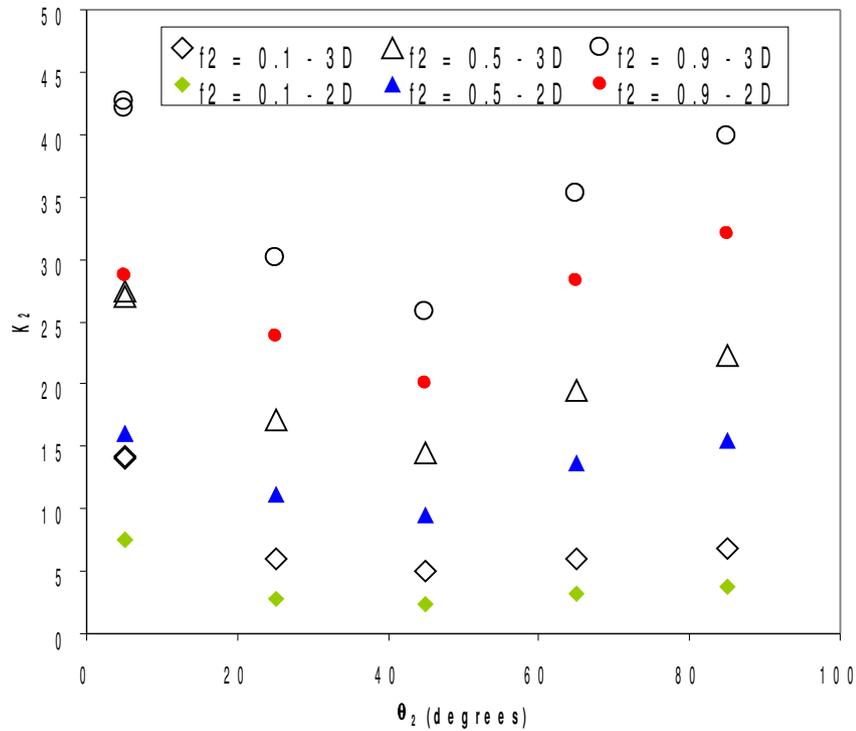
$f_2 = 0.7, \theta_2 = 65^\circ, \theta_3 = 45^\circ, d_2/d_1 = 0.5, d_3/d_1 = 1.5.$



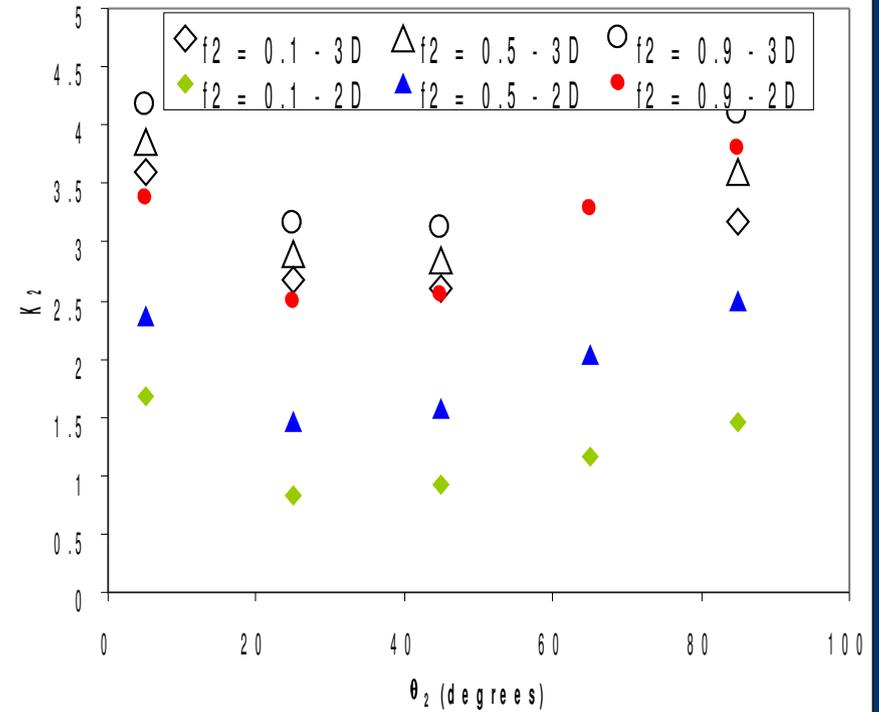
$$K_2 = 25.7$$

# 2D vs. 3D Differences

$K_2$  values for  $d_2/d_1 = 0.5$ ,  $d_3/d_1 = 0.5$ ,  $\theta_3 = 5$  degrees



$K_2$  values for  $d_2/d_1 = 0.5$ ,  $d_3/d_1 = 1.5$ ,  $\theta_3 = 45$  degrees



$$\left[ \frac{k_2^{3D}}{k_2^{2D}} \right]_{average} = 1.55 \quad (std. \ dev.) = 0.45$$

$$k_2 = \frac{\Delta p / \rho}{u_1^2 / 2} - \frac{u_2^2}{u_1^2} + 1$$

pressure effects
kinetic energy effects

# *3D Geometry Mark II - Junction*

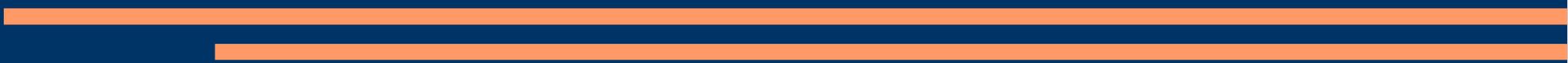
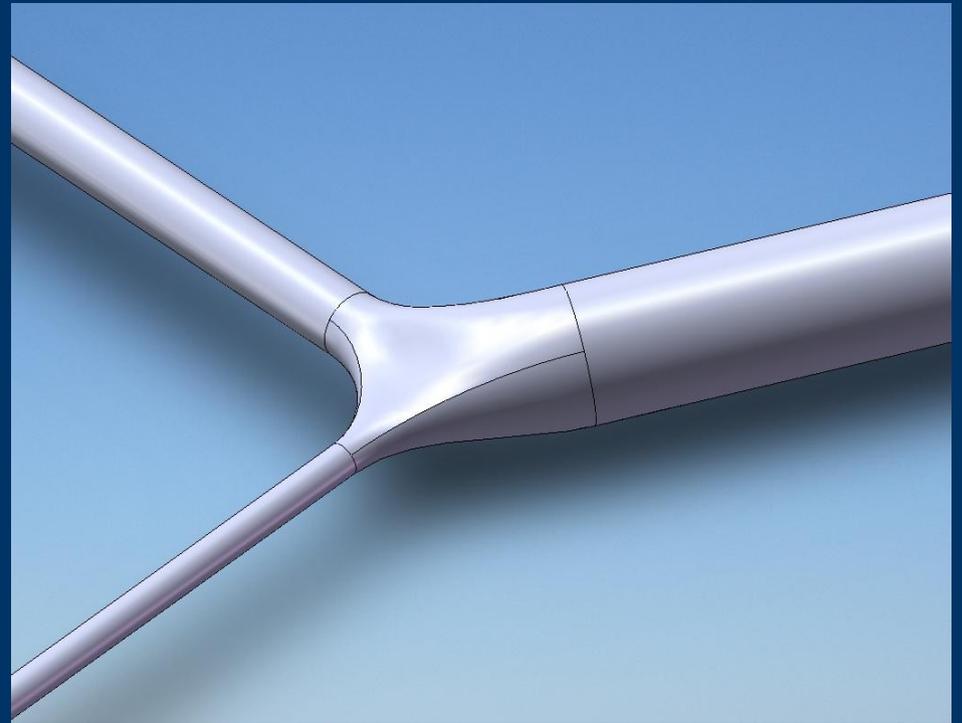
## *Algorithm*

No angle dependence

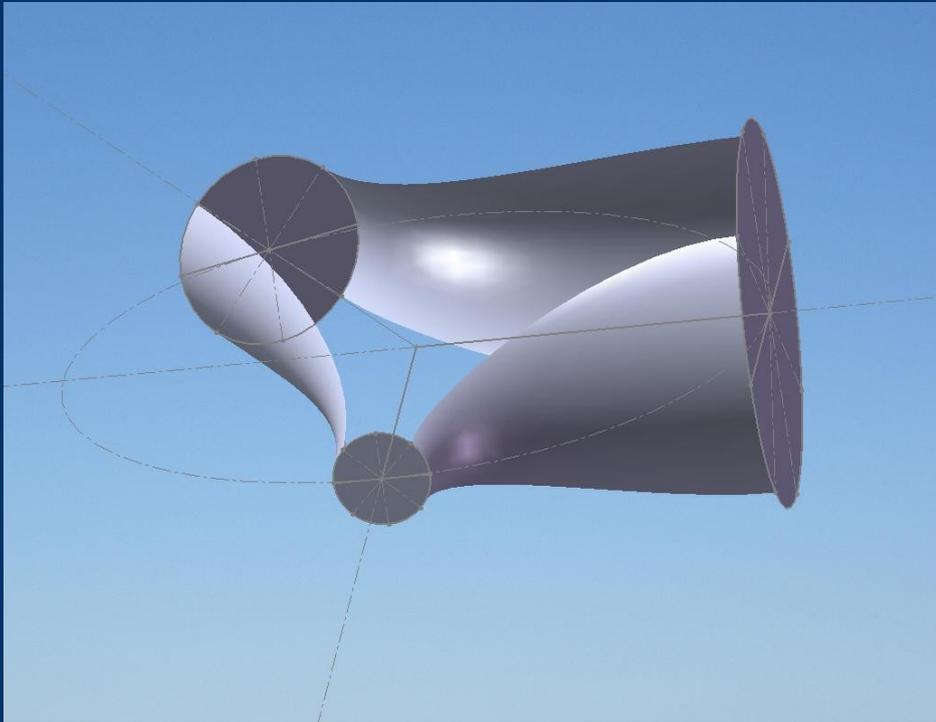
C++, VBA, SolidWorks,  
GAMBIT

Automated

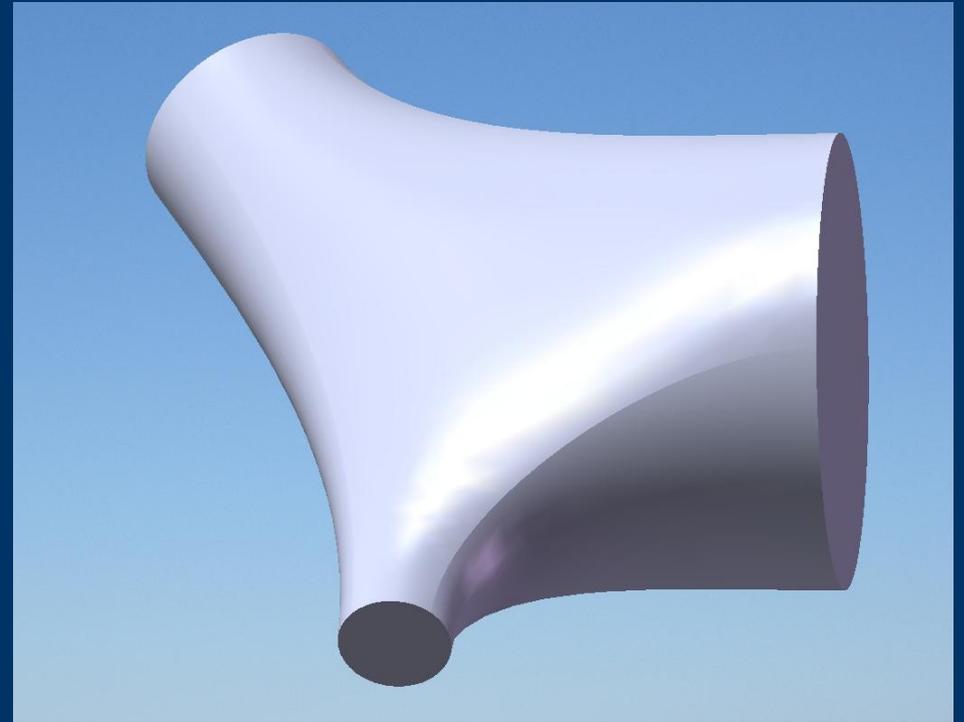
No failed geometries



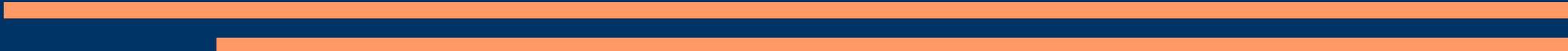
# *3D Mark II Junction*



Junction Without Surface Fills



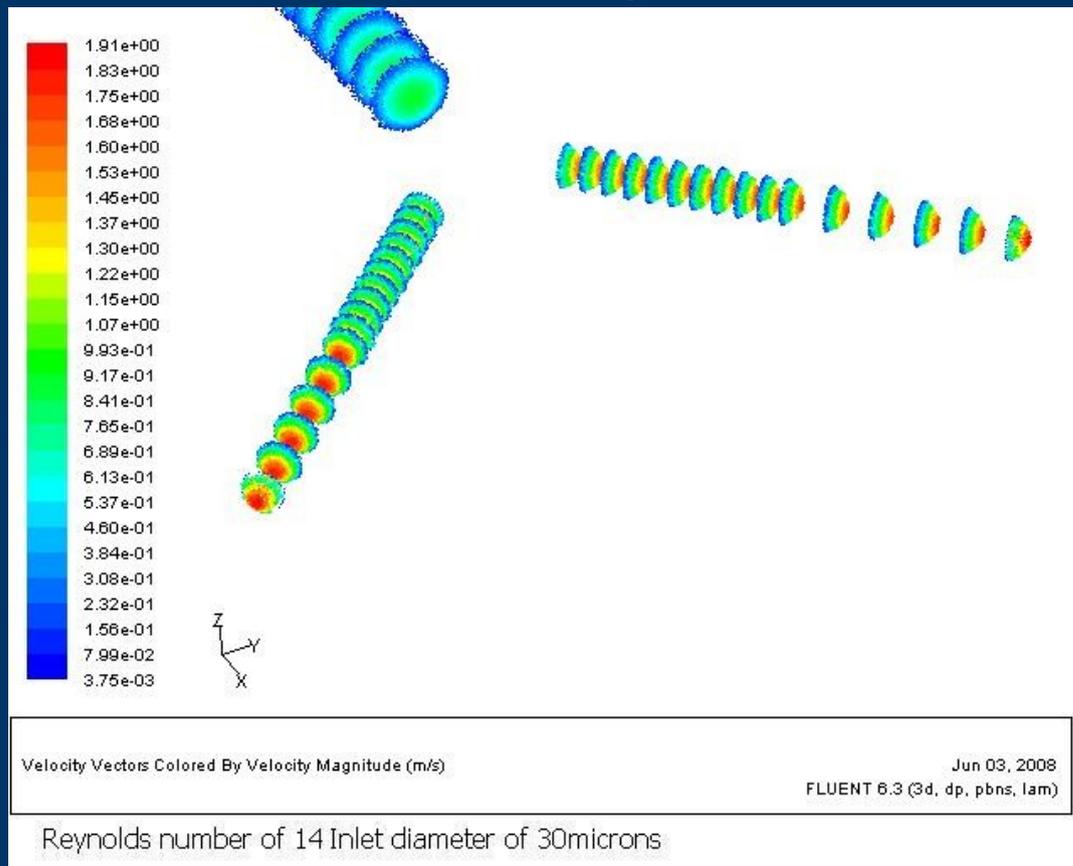
Junction With Surface Fills

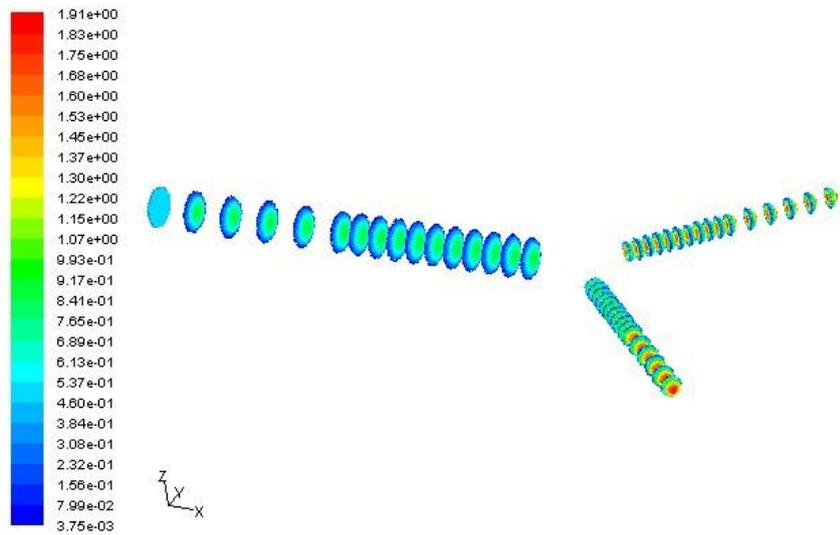


# Initial Simulations – 3D Mark II

Small batch

Comparable results to original 3D runs

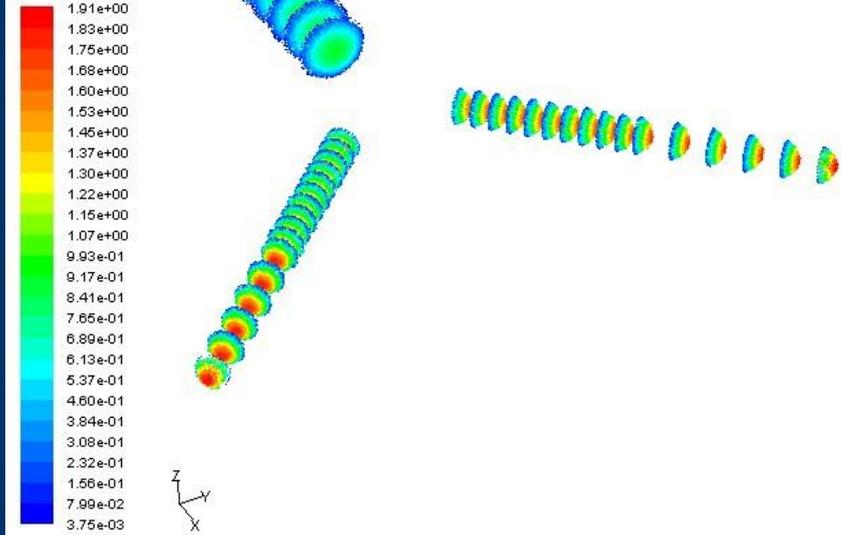




Velocity Vectors Colored By Velocity Magnitude (m/s)

Jun 03, 2008  
FLUENT 6.3 (3d, dp, pbns, lam)

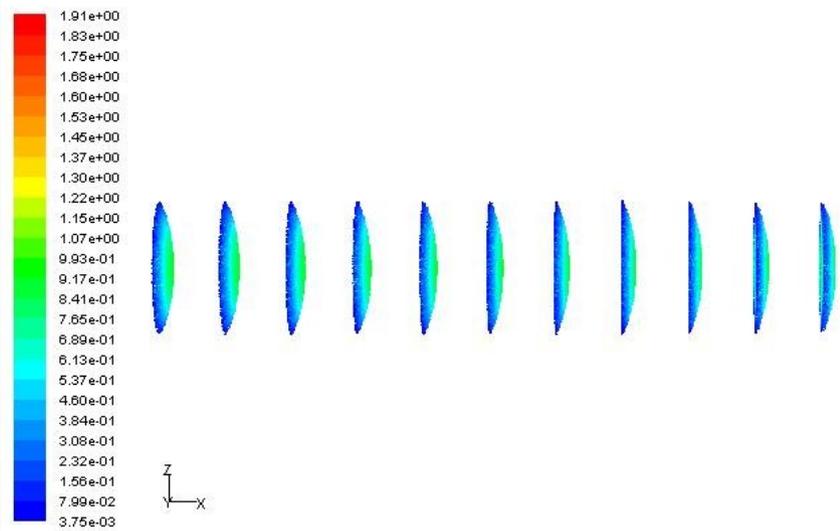
Reynolds number of 14 Inlet diameter of 30microns



Velocity Vectors Colored By Velocity Magnitude (m/s)

Jun 03, 2008  
FLUENT 6.3 (3d, dp, pbns, lam)

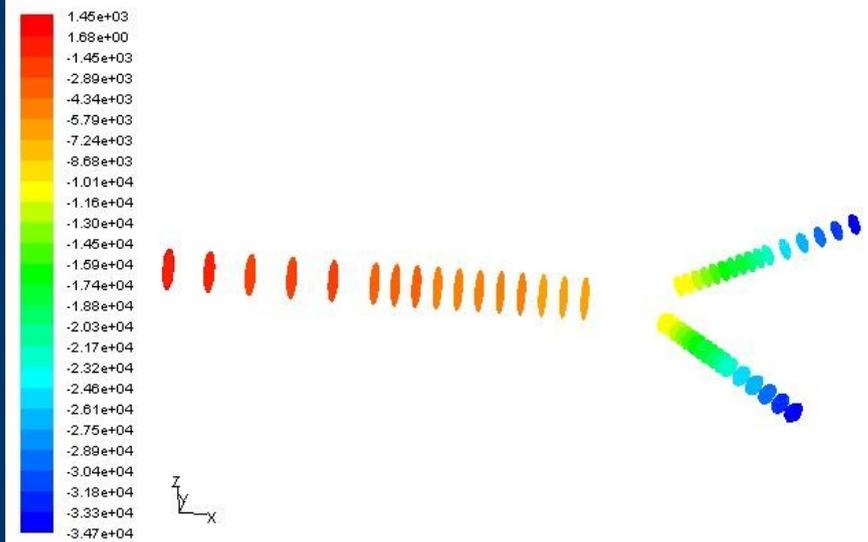
Reynolds number of 14 Inlet diameter of 30microns



Velocity Vectors Colored By Velocity Magnitude (m/s)

Jun 03, 2008  
FLUENT 6.3 (3d, dp, pbns, lam)

Reynolds number of 14 Inlet diameter of 30microns



Contours of Static Pressure (pascal)

Jun 03, 2008  
FLUENT 6.3 (3d, dp, pbns, lam)

Reynolds number of 14 Inlet diameter of 30microns

# *Pipe Mesh Testing*

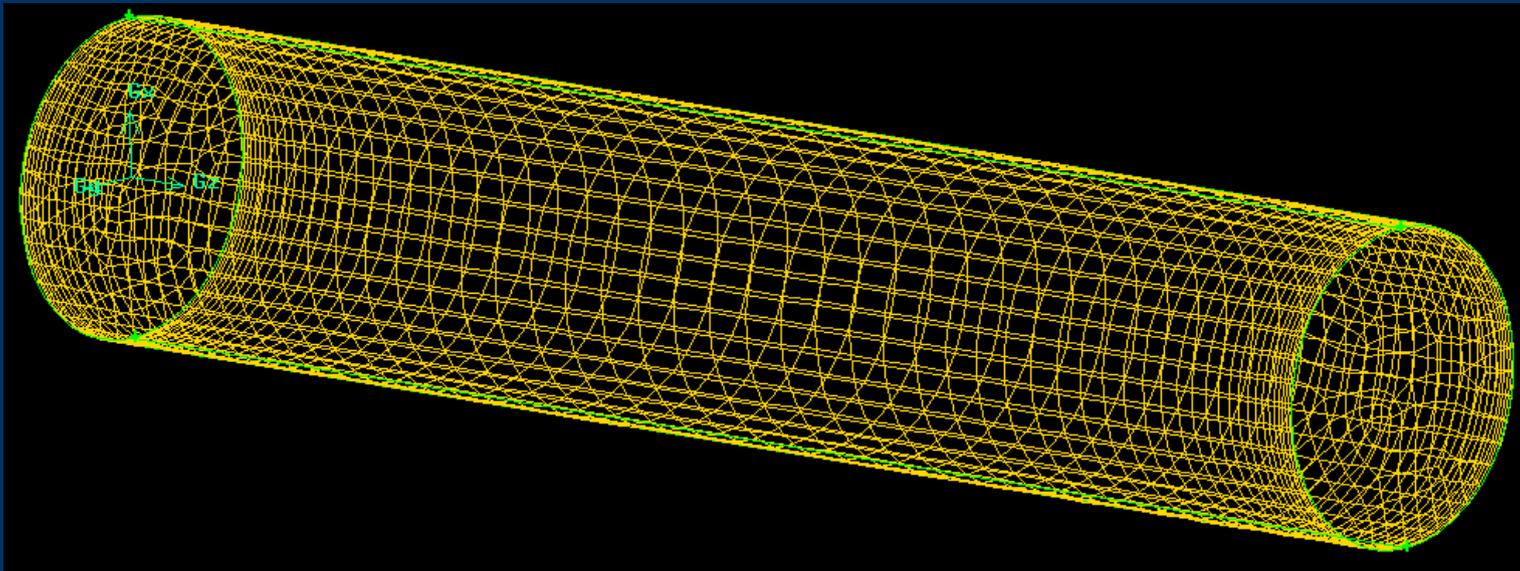
Flow-aligned hex core

Varied axial and radial spacing

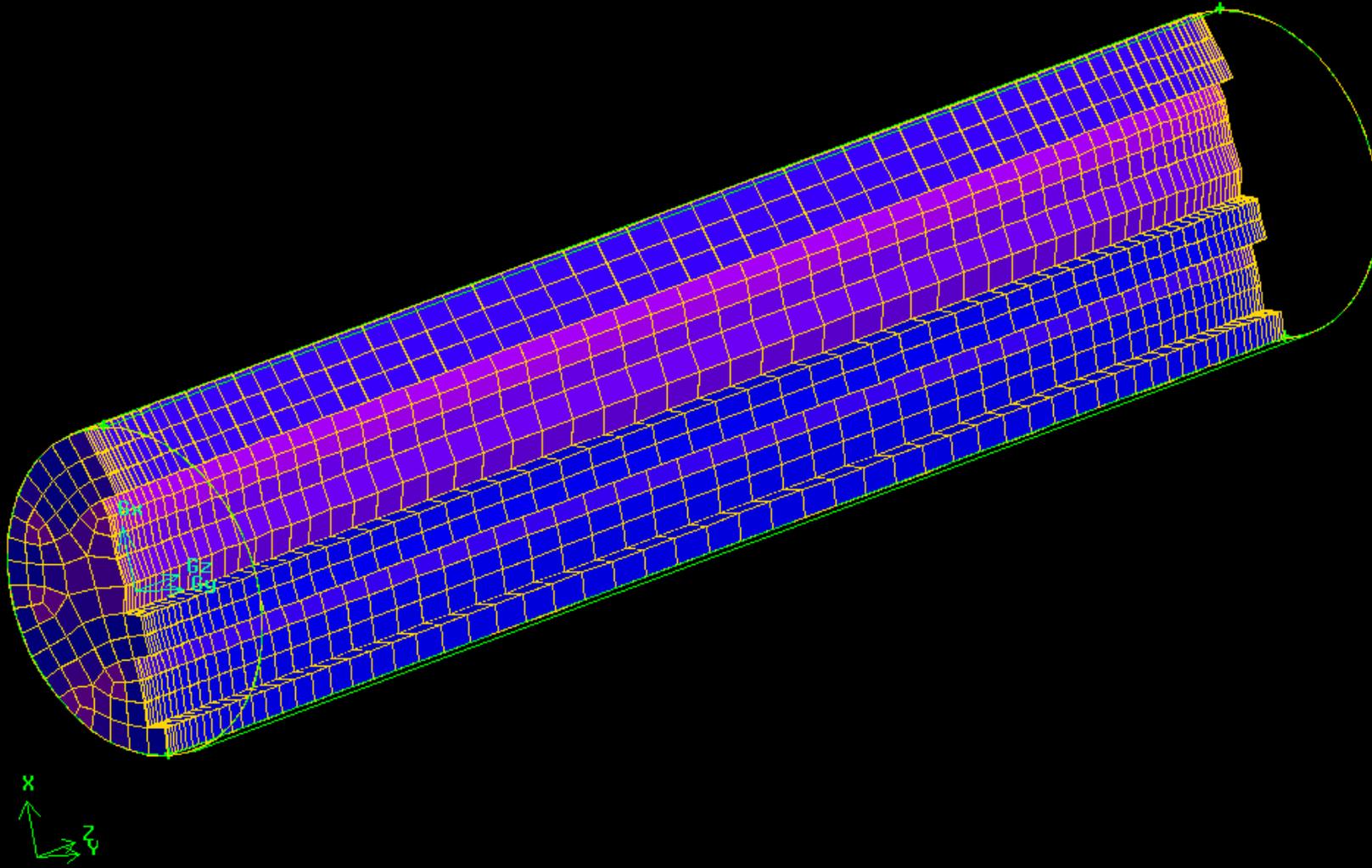
MySQL results database

Automated

Results database easy to setup for junction runs

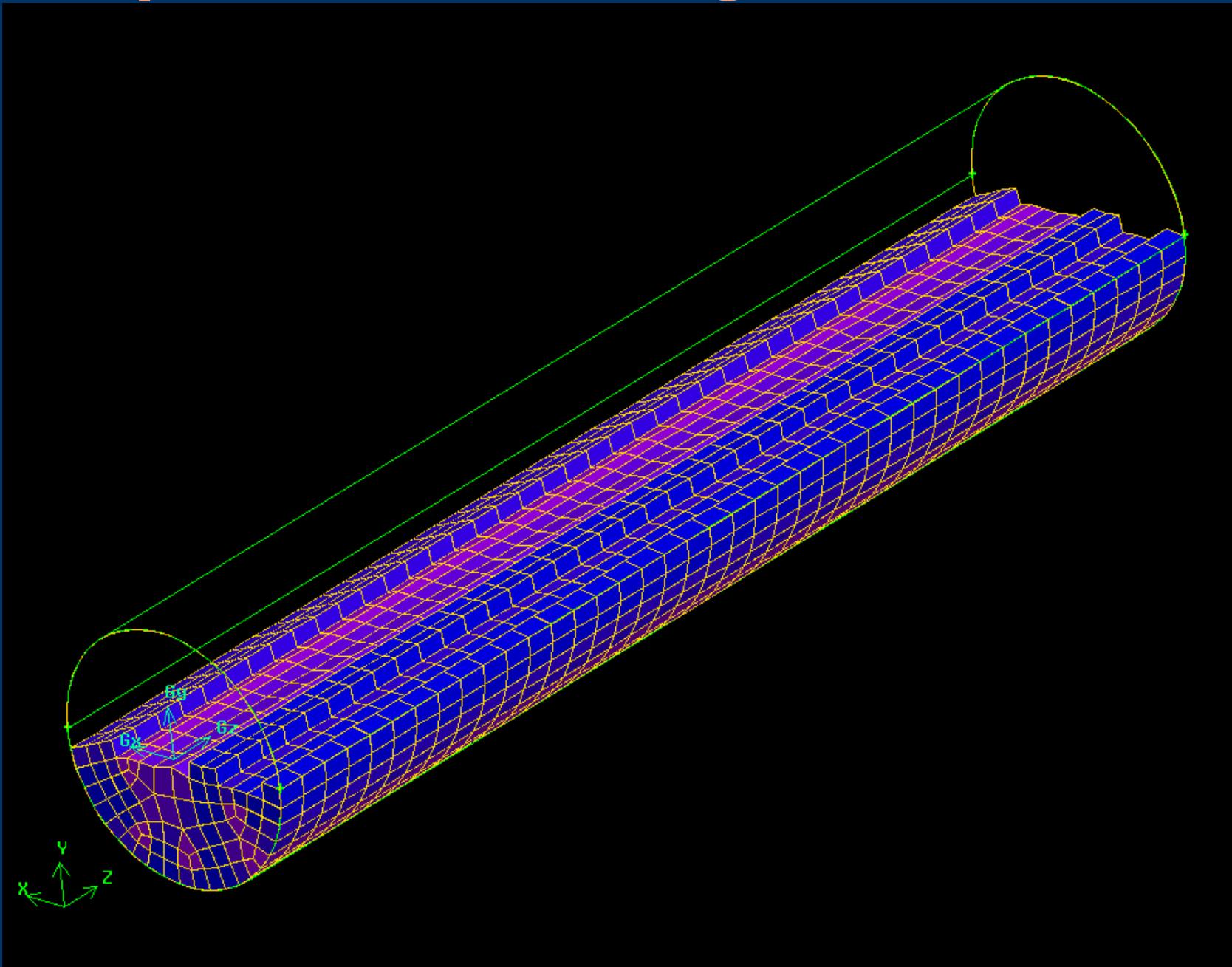


# Pipe Mesh Testing



**“Bell Shaped”  
Axial Spacing**

# Pipe Mesh Testing



Uniform Axial Spacing

# Pipe Mesh Testing

Database Interface

**Pipe Results Database**

Sever | Preview and Download | Open Runs

**Flow Parameters**

Reynolds: 25, 50, 100, 200, 500, 1000  
Diameters: 3  
Lengths: 30, 60, 150, 300, 450, 600

**Edgemesh Parameters**

Type: successive, bellshape  
Ratio1: 1, 0.8  
Spacing Type: 0  
Spacing: 10, 20, 50, 100, 30, 40

**Facemesh Parameters**

Type: map  
Spacing Type: 0  
Spacing: 20, 50, 70, 90, 40, 120

**Volumemesh Parameters**

Type: cooper  
Spacing Type: 1  
Spacing: 2

Get Valid Runs

**Valid Runs**

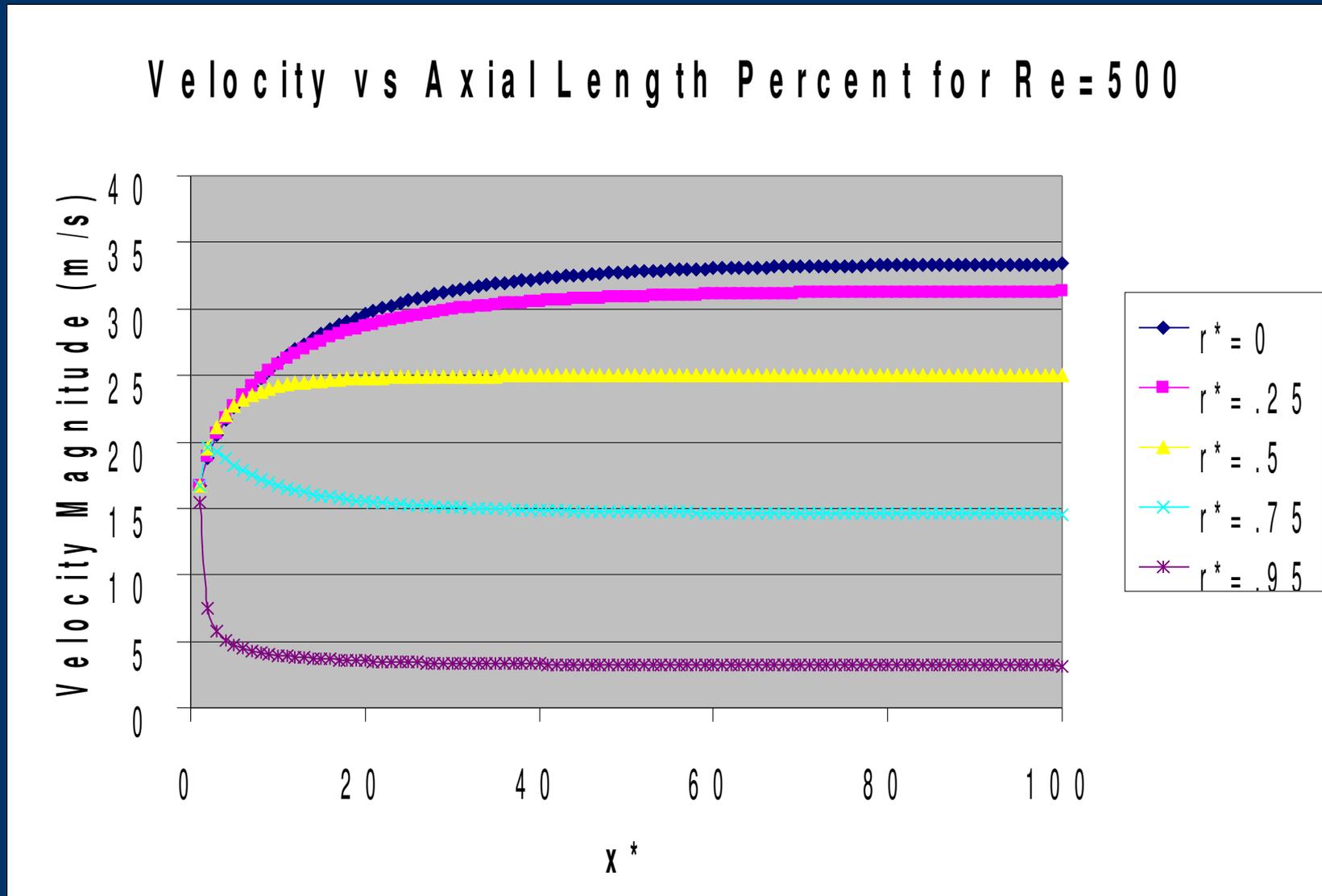
	Reynolds	Diameter	Length	Edge Params	Face Params	Volume Params
<input type="checkbox"/>	500	3	150	bellshape 0.8 0 80	map 0 90	cooper 1 2
<input type="checkbox"/>	500	3	150	bellshape 0.8 0 80	map 0 70	cooper 1 2
<input type="checkbox"/>	500	3	150	bellshape 0.8 0 80	map 0 50	cooper 1 2
<input type="checkbox"/>	500	3	150	bellshape 0.8 0 60	map 0 90	cooper 1 2
<input type="checkbox"/>	500	3	150	bellshape 0.8 0 60	map 0 70	cooper 1 2
<input type="checkbox"/>	500	3	150	bellshape 0.8 0 60	map 0 50	cooper 1 2
<input type="checkbox"/>	500	3	150	bellshape 0.8 0 50	map 0 90	cooper 1 2
<input type="checkbox"/>	500	3	150	bellshape 0.8 0 50	map 0 70	cooper 1 2
<input type="checkbox"/>	500	3	150	bellshape 0.8 0 50	map 0 50	cooper 1 2

Get Data | Plot Preview

**Rakes (Reynolds = 500)**

The graph displays the results of a mesh test at Reynolds = 500. The x-axis represents a normalized distance from 0 to 1.2, and the y-axis represents a value from 0 to 40. Multiple colored lines represent different mesh configurations. All lines start at approximately y=20 at x=0, rise to a peak between x=0.2 and x=0.4, and then level off. The highest peak is around y=35, while the lowest is around y=10.

# Single Tube Results for Entry Length ( $Re = 500$ )



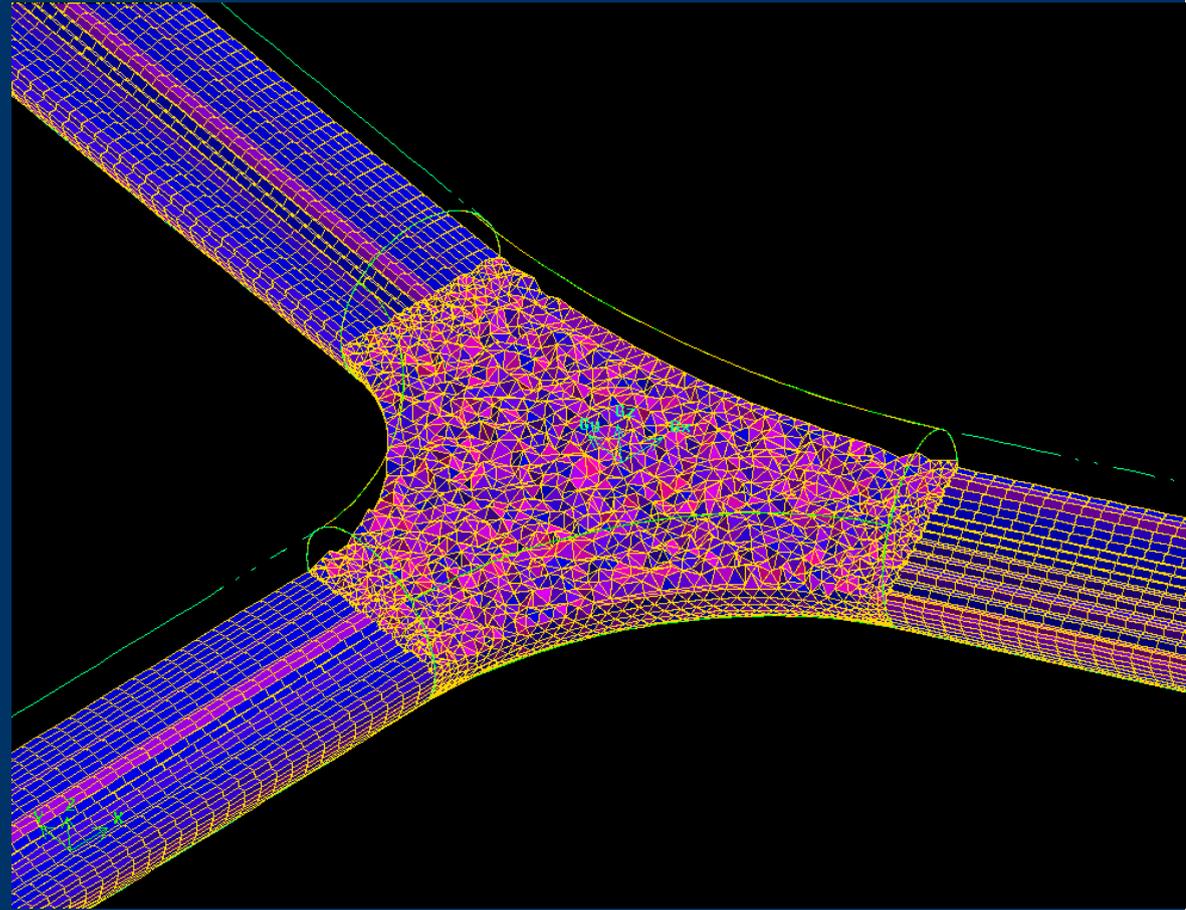
# *New Junction Simulations*

Combination of previous methods to allow for complete control and automation from geometry creation to data analysis

Flow-aligned hex core in pipes

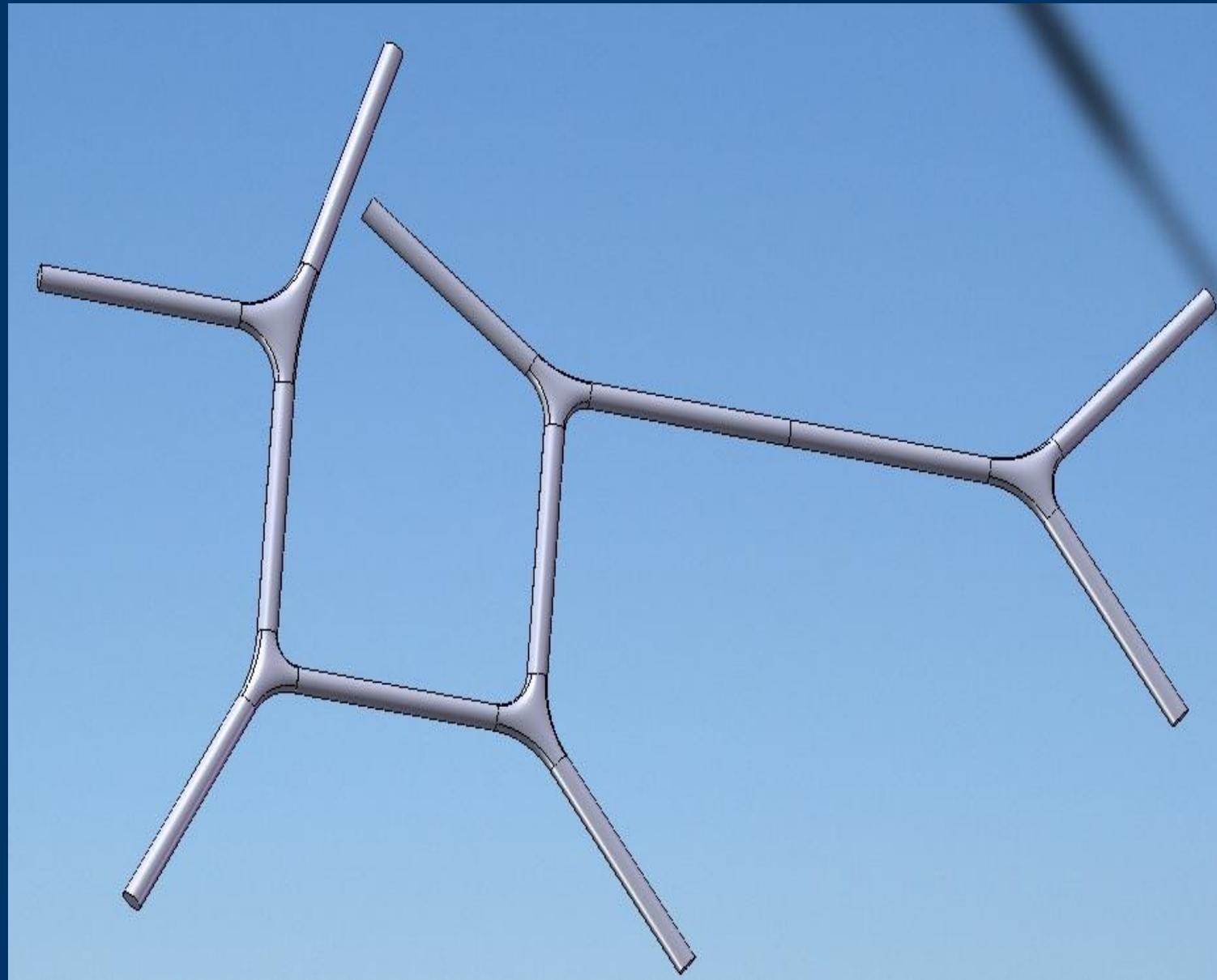
Tetrahedral elements in junction region – do not want to “guide” flow by using hex core

Use of HPC Solution to speed up cases and higher order meshing.



# Future Work

- Immediately able to plug into porous networks from Flow Through Porous Media (FTPM)
- Non-planar micro-bifurcations
- “N-furcations”
- Out of plane junctions (for use in porous media code)
- Effects of Roughness
- Experiments to establish laminar loss coefficients for number of configurations (in progress)



# *Acknowledgments*

The Office of Research and Grants at the University of Central Oklahoma is acknowledged for support of this research.

The Donors of The Petroleum Research Fund, administered by the American Chemical Society, are also be acknowledged for support of this research through grant PRF# 47193-B9.

National Science Foundation EPSCoR  
Research Opportunity Award Program

Dimitrios Papavassiliou and Henry Neeman

---

---